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MODELING OF CEILING FIRE SPREAD AND THERMAL RADIATION.(U)
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# MODELING OF CEILING FIRE SPREAD AND THERMAL RADIATION

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**FINAL REPORT** 

OCTOBER 1981

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Prepared for

U. S. DEPARTMENT OF TRANSPORTATION
FEDERAL AVIATION ADMINISTRATION
Systems Research & Development Service
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			Technical Report Documentation Page		
1. Report No.	2. Government Acces	sion No.	3, Recipient's Catalog No.		
DOT/FAA/CT-81-70	AD A1075	90			
4. Title and Subtitle			5. Report Date October 1981		
Modeling of Ceiling Fire	Spread and The	ermal	6. Performing Organization Code		
7. Author/s)		· · · · · · · · · · · · · · · · · · ·	6. Performing Organization Report No.		
R. L. Alpert, M. K. Mat	news, A. T. Mod	ak	OEON8.BU; RC81-BR-3		
9. Performing Organization Name and Addr Factory Mutual Research			10. Werk Unit No. (TRAIS)		
1151 Boston-Providence Norwood, Massachusetts		•	11. Contract or Grant No. DOT-FA79NA-6019		
12. Sponsoring Agency Name and Address			13. Type of Report and Period Covered		
U.S. Department of Tran. Federal Aviation Admini			May 25, 1979-March 1981		
Technical Center Atlantic City Airport,	New Jersev 084	05	14. Sponsoring Agency Code		
15. Supplementary Notes		- <u></u>	<u> </u>		
FAA Technical Center, f (NAFEC)	ormerly Nationa	l Aviation Fac	cilities Experimental Center		
16. Abstract  The pressure modeling technique is used to study fire spread under five differenceiling materials and analytical and numerical techniques are used to compute thermal radiation to floor level from the resultant layer of hot gases near the ceiling. In the physical modeling part of the study, measurements are obtained at one atmosphere (full-scale) and at elevated air pressure characterizing fire growth in a ceiling channel exposed to a developing PPMA wall fire. Pressure modeling predictions of flame spread rates under a PPMA ceiling and flame lengt under an inert ceiling are found to be in reasonable agreement with full-scale behavior. Although fire spread under aircraft material ceilings occurs only at elevated pressure and not at one atmosphere (due to charring effects and the use of full-scale material thickness in the models), exponential growth factors characterizing fire spread rates, mass loss rates and radiant heat loss in the model tests are used to group the five ceiling materials according to fire growth hazard. In the second phase of the study, an exact, numerical solution technique is formulated for computing the radiant flux from hot gas layers with arbitrary, three-dimensional variations in gas temperature and absorption coefficient. A simplified, analytic approximation involving the use of a suitable averaged gas temperature and absorption coefficient is compared with the exact technique for the calculation of radiant flux to targets below the ceiling gas layer. It is found that the analytic approximation is adequate even when gradients in temperature are much larger than those expected from real aircraft cabin fires.					
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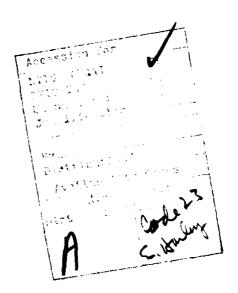
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#### PREFACE

This work was sponsored by the Federal Aviation Administration, through a contract from the FAA Technical Center, Atlantic City, New Jersey. The Contracting Officer's Technical Representative was Dr. Thor I. Eklund, whose valuable advice and encouragement during the entire course of this work are gratefully acknowledged.

Mr. Francis B. Kiley, aided by Mr. Lawney Crudup, Jr., ably performed the experiments described herein at the Factory Mutual Research Corporation. The support and patience of Dr. John de Ris, Manager, FMRC Basic Research and Dr. Raymond Friedman, Manager, FMRC Research Division, are sincerely appreciated.



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#### LIST OF SYMBOLS

- a width of parallelepiped section of ceiling layer
- b length of parallelepiped section of ceiling layer
- C specific heat of wall or ceiling material
- d flame thickness or gas layer depth
- H, height of lower edge of ceiling layer above target
- H, height of upper edge of ceiling layer above target
- k infrared absorption coefficient
- \_ radiation mean-beam-length
- l optical path length in ceiling layer
- m mass of wall or ceiling material
- m mass loss rate
- N radiance of ceiling layer for ray normal to ceiling
- n unit normal vector to target surface
- p absolute gas pressure
- d" heat flux or power per unit area
- radius vector from target along ray to far edge of ceiling layer
- T gas temperature
- t time
- V<sub>f</sub> flame spread rate
- W width of gas layer or channel
- $x_f$  flame length from end-wall
- x longitudinal Cartesian coordinate
- y lateral Cartesian coordinate
- z vertical Cartesian coordinate
- z vertical coordinate at far edge of ceiling layer
- $\mathbf{z}_{_{1}}$  vertical distance below ceiling surface
- a reradiation constant, between zero and unity
- γ exponential growth factor
- 0 polar angle
- $\lambda$  thermal conductivity
- o density
- o Stefan-Boltzmann constant
- φ azimuthal angle

#### Superscripts

' at absolute pressure, p

#### Subscripts

- c convective
- g gas
- lower limit of integration
- o reference or normal, one-atmosphere condition
- s ceiling or wall surface
- u upper limit of integration

#### I INTRODUCTION

#### 1.1 PURPOSE

The purpose of this project is 1) to model the fire growth which can occur along ceiling channels (simulating an aircraft fuselage) through the use of small-scale experiments at elevated air pressure and 2) to model the radiant heat transfer from ceiling fire gases in a long channel through analytical and numerical techniques.

#### 1.2 BACKGROUND

Fire growth within an aircraft cabin environment can occur by several different modes. One such mode is by upward fire spread on vertical surfaces, a process which can be very rapid. In a previous study<sup>1</sup>, the feasibility of modeling upward fire spread on vertical walls through the use of small-scale experiments at elevated air pressure (pressure modeling) was proven for a fuel with simple vaporization characteristics, polymethylmethacrylate (PMMA). Relative rates of upward spread among 15 different aircraft materials at elevated air pressures were also investigated in this study. As a result of the work in reference 1, it was found that samples of aircraft materials could be conveniently ranked by characteristic rate of upward fire spread, with such a ranking being reasonably independent of absolute air pressure in the range from 10 to 30 atmospheres.

Another mode for fire growth in a fuselage type of enclosure is by flame spread on the long ceiling channel formed by the cabin ceiling and walls. hot gas layer near the ceiling resulting from such fire spread can lead to full enclosure involvement in fire due to radiative heat transfer to lower levels at large distances from the ignition location. Fire spread under ceilings as well as steady burning of ceilings has not been studied in detail compared to other modes of fire spread. Recent progress in this area has resulted from work which appears in references 2-8. Babrauskas2 estimates flame lengths under open and channeled inert ceilings due to pool fire plume impingement. A simplified mass flow calculation procedure is used in reference 2 to obtain results for comparison with extensive measurements by You and Faeth' on smallscale (less than 0.34 m height) axisymmetric impinging flames and by Hinkley et al 11 on flames along large-scale channels. Flame spread in the direction of forced convective flows is analyzed by Fernandez-Pello<sup>6</sup> and by Annamalai and Sibulkin for the laminar case where thermal radiation is unimportant and by Carrier et  $a1^5$  for the tunnel geometry used in the ASTM E84 flammability test. In the latter reference<sup>5</sup>, numerical solution of a very complex mathematical formulation is required. Movement of smoke or combustion products in largescale ceiling channels has been analyzed recently by Delichatsios<sup>4</sup> while Hwang et al $^9$  have previously studied product flow along the ceilings of ventilated ducts. Work done by Alpert $^{10}$  establishes simplified integral models for nonreacting ceiling-jets in two-dimensional channels. The flow studied in

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reference 10 results from impingement of a line plume across the entire channel width rather than from axisymmetric plume impingement, as in reference 4. None of the preceding theoretical work directly addresses the problem of predicting self-sustained fire spread in ceiling channels, although many of the results can eventually be applied to making such predictions.

Because of the absence of proven theoretical predictive techniques, the first part of the present study investigates the possibility of physical modeling of the ceiling fire spread phenomenon through use of the pressure modeling technique. In this first section of the report, small-scale tests run at elevated ambient pressure yield measurements of transient flame lengths, mass loss rates and radiant heat loss for five different ceiling materials exposed to a PMMA wall fire at one end of a long channel. Full-scale tests with the same five materials and an analysis of the effect of pressure on radiant heat loss are used to evaluate modeling success. Results show the modeling to be surprisingly accurate for both inert and PMMA ceilings in spite of relatively thick hot-gas layers, radiation from which cannot be readily modeled at elevated air pressure. However, it is shown that the flame spread process is not modeled for three aircraft material ceilings, probably due to charring effects and the impracticality of changing the thickness of complex, composite materials. (A factor of 10 decrease in full-scale thickness is required for modeling at 31 atmospheres.)

In the second part of the report, radiant heat loss from realistic, corridor-like ceiling fires is examined in detail. As part of this study, the consistency of several different ceiling layer temperature and heat flux measurements made during the series of full-scale tests is analyzed. Exact numerical calculation of radiant heat flux from hot gases in a long ceiling channel (such as an aircraft cabin) are then shown to be in very good agreement with results from an approximate, analytical technique. This analytical procedure greatly simplifies the task of calculating radiant emission from inhomogeneous hot gas lavers of known size, composition and temperature.

#### 2.1 EXPERIMENTAL ARRANGEMENT

- 2.1.1 MODELING TECHNIQUE. The pressure modeling technique for prediction of transient, large-scale fire phenomena from small-scale experiments is fully described in references 12-14. As explained in these references, the modeling scheme requires the reduction of all characteristic length scales as the minus 2/3 power of absolute air pressure. This type of scaling preserves the full-scale relationship among gas phase inertial, buoyant and viscous forces. Solid phase thermal response and vaporization characteristics will also model full-scale phenomena, but on a time scale which is reduced as the minus 4/3 power of absolute air pressure. The previous study of upward fire spread has shown that the modeling scheme may not generally be successful if thermal radiation from solid surfaces or from thick gas layers is the dominant heat transfer mode in fire growth.
- 2.1.2 WALL-CHANNEL CONFIGURATION. The prototype configuration to be pressure modeled is illustrated by the schematics in Figures 1 and 2. A 2.4 m long, 0.46 m wide channel is formed by a replaceable ceiling and two permanent sidewalls extending 0.23 m below the ceiling. At one end of the channel is a 0.9 m high PMMA wall separated from the ceiling by a 0.32 m high inert wall section.

Tests are initiated by ignition of the PMMA wall at the single point shown in the schematics. Use of a point ignition greatly enhances experimental reproducibility and simplifies the elevated pressure model tests. About 16 to 17 minutes after this ignition, there is steady flame impingement on the ceiling material and about 25 to 27 minutes after ignition, flames have spread laterally to almost the full, 0.305 m width of the PMMA wall. If the PMMA wall width were equal to the 0.46 m channel width, complete lateral flame spread would not occur during the time available for the experiment due to the point ignition and very low lateral flame spread rate. Some 35 to 40 minutes after ignition, safety considerations generally dictate a termination of the test by extinguishment of the PMMA wall fire.

The height and position of the PMMA wall and the depth of the channel and vertical side walls that are shown in Figures 1 and 2 were selected after a number of model experiments at elevated air pressure. These 20-atmosphere tests indicated that complete flame spread over an inert ceiling would occur within 30 minutes (at full-scale) solely as a result of the PMMA wall fire if a 1.22 m high wall in direct contact with the ceiling were used, as originally planned. In order to distinguish among different combustible ceilings on the basis of flame spread rate, the PMMA wall height was reduced to 0.9 m; a 0.32 m high inert wall was inserted between the top of the PMMA wall and the ceiling. With this final arrangement, it was found from the model experiments that flames from the PMMA wall fire would not extend along the complete length of an inert ceiling for the equivalent of more than 30-40 minutes (see Sections 2.3 and 2.4).

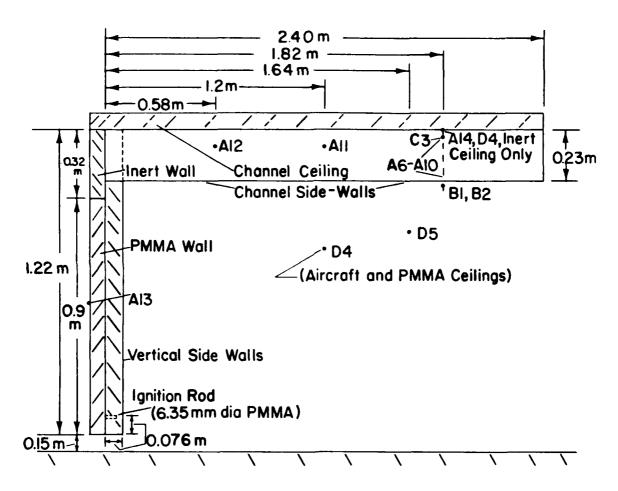


FIGURE 1 SIDE VIEW OF CHANNEL

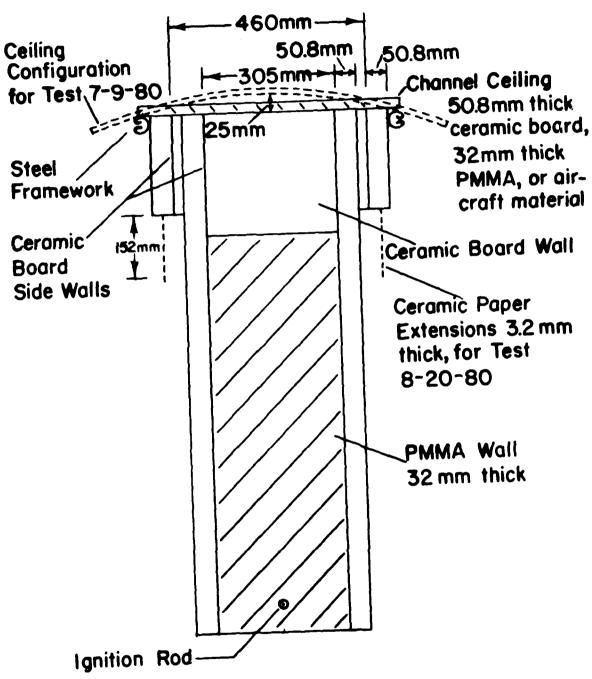


FIGURE 2 END VIEW OF CHANNEL

Model experiments were also used to determine the minimum channel depth (distance from ceiling to bottom of channel side-wall) required to contain the ceiling gas flow. Initial experiments with 0.15 m channel depth revealed an excessive amount of gas leakage with inert and combustible ceilings. Such leakage was virtually eliminated with the final selected depth of 0.23 m. This value is in good agreement with the predicted maximum thickness of a two-dimensional (planar) ceiling-jet, as obtained from the theory of reference 10 (Figure 6). For a ceiling height above the "point" fire source of 0.77 m (assume the point source is the mid-height of the PMMA wall) and a ceiling-jet length of 2.4 m, the predicted value of ceiling-jet thickness from reference 10 is 0.21 m.

The depth of the vertical, confining side-walls on the PMMA wall at the end of the channel were also selected from model experiments. It was found from model tests that a vertical side-wall depth of 0.05 m was inadequate for containing flames from the PMMA wall at a height of 1.22 m above the wall base. The minimum acceptable side-wall depth of 0.076 m, finally chosen, represents 1/16 of the total PMMA and inert wall height, a ratio obtained previously 15 from full-scale tests.

2.1.3 FULL-SCALE TESTS. Full-scale experiments are performed with each of the five ceiling materials listed in Table 1. In addition, duplicate tests are run with the inert and PMMA ceilings for much longer durations than the initial tests. For the PMMA experiment, data are only obtained during a steady burning period of full ceiling involvement.

Each of the seven full-scale tests are instrumented according to the schematic in Figure 1, which contains instrument symbols explained in Table 2. All instrumentation is located as close as possible to the mid-plane of the channel. The main measurement station for these experiments is about 1.82 m from the PMMA wall. At this station are located two different narrow angle radiometers viewing the bottom of the smoke layer, an array of thermocouples, and a smoke absorption meter consisting of a radiometer viewing an optically-chopped, 725 C blackbody oven. Both narrow-angle radiometers utilize an optically chopped beam, amplifiers "locked-in" to the chopping frequency and right angle mirrors for viewing the layer with a horizontal radiometer axis. Calibration of the radiometers is performed with right angle mirrors installed. In addition, a thermocouple and a water-cooled Gardon-type heat flux gage are mounted flush with the surface of the inert ceiling in order to obtain surface temperature and total heat flux to the ceiling respectively. With combustible ceilings, the surface thermocouple is not used. Instead, a (180°) wide angle, radiant flux measurement is made by an upward facing Gardon gage at the center of the channel, 0.763 m below the ceiling surface.

Gas temperature distributions are obtained from thermocouples at the main measurement station and at three other locations. Radiation errors are minimized by use of 0.127 mm dia chromel-alumel wire and by thermocouple bead sizes which are less than twice the wire dimension. Conduction errors are also minimized by keeping thermocouple wires coincident with expected isotherms.

TABLE 1.-DESCRIPTION OF CEILING MATERIALS

Ceiling Material	Material Composition	Material Thickness
Inert	"Cotronics" type 360	$50.8 \text{ mm}: p/p_0 = 1.0$
	ceramic board	12.7 mm: $p/p_0 = 11.2$
		6.35 mm: $p/p_0 = 21.4$
		and $p/p_0 = 31.6$
P <b>MMA</b>	Polymethyl methacrylate	31.75 mm: p/p <sub>o</sub> = 1 0
	cast sheet ("Plexiglas"	6.35 mm: $p/p_0 = 11.2$
	type G)	3.18 mm: $p/p_0 = 21.4$
		and $p/p_0 = 31.6$
No. B8811	Honeycomb Composite	6.99 mm overall
	(solid face 0.64 mm thick)	
No. 223	Honeycomb Composite	12.45 mm overall
	(perforated face, 0.38 mm	
	thick, melts and sags, expos-	-
	ing honeycomb during fire sp	read)
No. 234	Fiberglass Reinforced	2.03 mm
	polyester (rigid solid	
	sheet)	

TABLE 2.-MEASUREMENT INSTRUMENTS FOR FULL-SCALE TESTS

Instrument I.D.	Description
В1	Radiometer, 1.3° total view angle of smoke layer, optically chopped, orifice 0.23 m below* ceiling, sensor 0.79 m from ceiling (also used for pressure modeling measurements).
В2	Radiometer, 25.4 mm diameter parallel beam from smoke layer, optically chopped, orifice 0.23 m below ceiling. Uses nitrogen purged, right angle mirror.
С3	Smoke Absorption Radiometer. 25.4 mm diameter parallel beam views optically chopped radiation from 725°C oven, optical axis 70 mm below ceiling, 300 mm path length for smoke absorption. Nitrogen purged, honeycomb view ports define absorption path length.
D4	Gardon-type, Total Heat Flux Gauge, 180° view angle, 0-10 W/cm² range, flush with surface of inert ceiling (facing down), 0.763 m below aircraft and PMMA ceilings (facing ceiling).
D5	Gardon-type Radiometer, 0-10 W/cm <sup>2</sup> range, Nitrogen-purged Irtran II window, 90° total view angle of smoke layer, 0.61 m below ceiling.
<b>A</b> 6	Thermocouple in smoke layer, 30 mm below ceiling
A7	Thermocouple in smoke layer, 63 mm below ceiling
A8	Thermocouple in smoke layer, 130 mm below ceiling
А9	Thermocouple in smoke layer, 191 mm below ceiling
A10	Thermocouple in smoke layer, 225 mm below ceiling
A11	Thermocouple in smoke layer, 70 mm below ceiling
A12	Thermocouple in smoke layer, 64 mm below ceiling
A13	Thermocouple on back surface, center of PMMA wall
A14	Thermocouple on surface of inert ceiling.

<sup>\*</sup>Increase all below-ceiling distances by 25 mm (see Figure 2) for Test 7-9-80.
All instrumentation is mounted nearly equidistant from the two channel side-walls.

The flame spread process on the vertical PMMA wall and under the ceiling is photographed with two 35 mm cameras. One camera, located behind the transparent PMMA wall, views the advance up the wall of flame followed by the onset of surface bubbles which characterize the beginning of fuel pyrolysis. A clear view of this pyrolysis front and the upward spread process is obtained from behind the wall if the camera is in darkness and an illuminated white surface in front of the wall is used as a background. The second camera, which is motorized and equipped with a 250-exposure-capacity film back, views the ceiling from the side of the channel structure through a long, back surface mirror set at a shallow angle (about 25° to the horizontal) on the laboratory floor. Because the mirror has roughly the same dimensions as the ceiling, the side camera records flame lengths and position along the entire ceiling surface and on much of the upper portion of the vertical PMMA wall. A digital clock and a length grid appear in each camera view to facilitate data reduction and synchronization of photographs with all other measurements.

The entire full-scale ceiling channel is located under a large-capacity, water-cooled combustion hood. During the initial stages of the test, when upward fire spread on the PMMA wall is in progress, combustion products exit the end of the channel and flow up to the hood exhaust duct by natural convection. A fan in the hood exhaust duct is activated once the capacity of the hood is taxed by the volume of combustion products. This generally occurs well after flame impingement on the ceiling material.

2.1.4 MODEL TESTS. Experiments with geometrically scaled models of the full-scale ceiling channel are performed at elevated air pressures of 11.2, 21.4 and 31.6 atmospheres. These experiments are initiated in a facility consisting of a pressure vessel of 1.22 m inside diameter and 2.7 m<sup>3</sup> volume. Combustion products are well above the model ceiling channel for all experiments due to stratification and an adequate vessel interior volume. Further details about the pressure vessel and the remainder of the facility at the Factory Mutual Research Corporation (FMRC) may be found in reference 16.

The configuration and composition of the wall and ceiling channel shown in Figures 1 and 2 are also used for the model experiments. All full-scale dimensions given in Figures 1 and 2 are reduced by the factor,  $(p/p_0)^{-2/3}$ , where p is absolute air pressure and  $p_0$  the reference, atmospheric pressure. One exception to this scaling rule is the thickness of the three aircraft material ceilings. Changing the thickness of complex, composite materials is generally not possible without total refabrication. As a result, the normal, full-scale aircraft material thickness is employed at all test pressures. However, the thickness of the inert and PMMA ceilings is scaled roughly as the required  $p^{-2/3}$ . Both these ceilings are thermally thick for the duration of the fire so that the exact ceiling thickness is unimportant.

Another exception to the geometric scaling at elevated pressures is the ignition mode shown in Figures 1 and 2. For all experiments at elevated pressure, a small wooden toothpick is used to ignite the PMMA wall, instead of a

scaled, PMMA rod. The energy applied by the burning toothpick or PMMA rod to the PMMA wall at all scales is probably close to the minimum amount needed for initiation of upward fire spread.

Instrumentation for the model experiments is necessarily limited by the space available in the vessel and the elevated pressure environment. As in the full-scale experiments, the flame position under the ceiling and the radiance of the bottom of the ceiling smoke layer are measured during the elevated pressure tests. In addition, the total mass loss rate of the combined wall-ceiling configuration is measured continuously by a load transducer coupled to the platform on which the ceiling channel is mounted.

The radiometer used to measure the radiance of the ceiling flames and ceiling surface for the pressure vessel experiments is fully described in reference 1. This instrument, which is also used for the full-scale tests, views the ceiling through a first surface, right-angle mirror. Although soot from the burning ceiling can be deposited on the mirror during a given experiment, cleaning of the mirror with acetone and cotton after each test prevents significant buildup of residue. Calibration of the radiometer-mirror system with a blackbody oven has shown that such mirror cleaning does not adversely affect infrared reflectivity.

Flame position under the model ceilings is obtained by a motorized 35 mm camera viewing the ceiling through a pressure vessel window and a long back surface mirror set at a shallow angle on the load platform. Because of limited space under the model ceiling, the viewing mirror only extends from the PMMA wall to near the right-angle mirror of the radiometer. Flame position measurements are thus limited to about 2/3 the total channel length, except that the existence of flame at the end of the channel can be viewed directly by the camera for the two smallest model structures used at 21.4 and 31.6 atmospheres.

#### 2.2 EXPERIMENTAL RESULTS: FULL-SCALE TESTS

Measurements from all seven full-scale tests are tabulated in the Appendix, generally as a function of time (in seconds) after ignition of the PMMA wall. The tabulated measurements can be used in conjunction with the schematics of Figures 1 and 2 and with the instrument descriptions in Table 2.

2.2.1 UPWARD FLAME SPREAD ON PMMA WALL. Measurements of flame height and pyrolysis height above the ignition point on the PMMA wall are given in Tables A-1 and A-2. These data represent the results of two separate experiments with an inert ceiling on the channel. Although the flame and pyrolysis fronts reach the top of the PMMA wall in about 750 and 900 s, respectively, the tear-shaped flame zone does not occupy the full,  $0.305 \, \mathrm{m}$  width of the PMMA wall at any height until roughly  $1500 \, \mathrm{s}$  after ignition.

A least squares, exponential regression fit to the upward spread data yields consistent, pyrolysis height growth factors for the two experiments but

significantly different flame height growth factors. The inconsistency in the latter case is likely due to rapid flame height fluctuations and the indistinct nature of the flame tip in photographs. In both cases, results compare favorably with measurements made in the previous study for a 1.83 m high PMMA wall in the open: average exponential growth factors for pyrolysis and flame heights of  $0.0027~\rm s^{-1}$  and  $0.0026~\rm s^{-1}$ , respectively, in the present study and  $0.0024~\rm s^{-1}$  and  $0.0023~\rm s^{-1}$  in the previous study. The somewhat higher growth factors for a PMMA wall beneath a ceiling channel could be due to radiative feedback from the ceiling, heated by combustion products in the channel flow.

- 2.2.2 MASS LOSS RATE OF PMMA WALL. An average mass loss rate per unit area of the burning PMMA wall is obtained from the known burning time at a given wall height (the extinguishment time after ignition minus the pyrolysis spread time to that height minus an 80 s transient (see reference 15) before quasi-steady burning), from the weight of burned wall segments cut along the vertical wall centerline after the test, and from the initial wall thickness. The resultant, centerline fuel mass flux is given in Table A-3 for the two experiments with an inert ceiling on the channel. As expected, average fuel mass flux is much higher for the longer duration test because of the gradual buildup of thermal radiation to the wall from both the channel hot gas layer and the channel ceiling. Thermal radiation from the ceiling channel is also the reason that the mass fluxes above a 0.6 m height for the shorter duration test are about 25% higher and for the longer duration test about 45% higher than those measured by Orloff et al 15 for a PMMA wall in the open. Below a 0.6 m wall height, fuel mass flux for the shorter duration test is about the same as that given in reference 15 but, for the longer duration test, about 24% higher than the reference 15 values.
- 2.2.3 <u>CEILING FIRE GROWTH</u>. The extent of flames under each tested ceiling of the channel structure is given in Figure 3. Flame lengths, also listed in Tables  $\Lambda$ -4 to  $\Lambda$ -9, refer to the horizontal distance between the inert section of the channel end wall (above the PMMA) and the average position of the flame tip. Segments of flame which are detached from the main body of visible flame are disregarded in making the flame length determination.
- 2.2.3.1 Inert Ceiling. With an inert, ceramic board ceiling on the channel, flames from the PMMA wall fire gradually increase in length under the ceiling due to lateral fire spread on the PMMA wall. Such spread is essentially complete by 25 minutes after ignition, when flames extend out 30-40 cm along the ceiling. Subsequent flame lengthening to about 1 m some 42 minutes after ignition (see Figure 3) is due to the gradual increase in temperature of the ceiling surface, with a consequent increase in radiative flux to the wall fire. This thermal radiation feedback from the ceiling surface and ceiling layer gases leads to higher wall burning rates and greater flame lengths as shown in Section 3.3.3. Although flame lengths are still increasing at 44 minutes after ignition due to this coupling of the wall fire with ceiling-induced thermal radiation, data on ceiling layer radiative properties in Table A-11 indicate that a quasi-steady condition has been achieved.

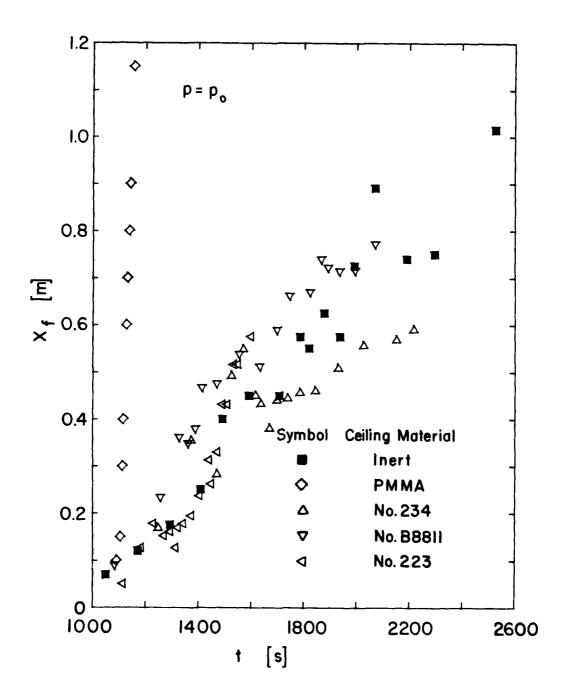


FIGURE 3 FLAME LENGTHS UNDER FULL-SCALE CEILINGS

During fire development, the white, ceramic board side-walls of the 2.4 m long ceiling channel are blackened wherever there is contact with hot combustion products or flame. Such markings clearly indicate a characteristic thickness of the hot gas layer in the channel. Measurements of the blackened zone dimensions on the channel side walls after the tests with an inert ceiling show layer thickness to be a nearly constant 178 mm from near the inert end wall (above the PMMA) to within 610 mm of the outflow end of the channel. Closer to the channel outflow end, these measurements show the layer thickness decreasing as follows:

Distance	from	End	[ mm ]	Layer	Thickness	{ mm }
	610				178	
	305				154	
	178				132	
	0				88	

2.2.3.2 Aircraft Material No. 234. This ceiling consists of five curved panels of fiberglass-reinforced polyester, oriented as shown in the Figure 2 schematic. No backing or insulation is applied to the panels. The horizontal width of the ceiling exposed to flame is maintained at the normal 460 mm through the use of ceramic sealants and gaskets which are also used to close any gaps between adjacent panels. Maximum ceiling height above the top of the PMMA wall is 25 mm greater than that for the other ceilings, due to the panel curvature.

During fire impingement on the ceiling panels, pyrolysis products are observed above the ceiling, presumably from decomposition of the ceiling material. This ceiling vaporization is obvious by 18.5 minutes after ignition and by 20 minutes after ignition, decomposition has resulted in buckling of the ceiling panel closest to the PMMA wall fire. However, flame spread due to ceiling combustion is not evident. As shown in Figure 3, flame lengths at equivalent times are comparable to those for the inert ceiling. There is some shortening of flames, in fact, due to the slightly increased ceiling height of the curved panels.

2.2.3.3 Aircraft Material No. B8811. This flat ceiling consists of a solid-faced, honeycomb composite. No backing or insulation is applied above the ceiling. Ceramic paste seals the single joint between panels, about 2 m from the wall fire. As a result of fire impingement, the ceiling blackens and blisters significantly but there is no flame spread due to ceiling combustion. Nearly the entire length of the ceiling is blackened and blistered by about 22 to 25 minutes after ignition, while flaming combustion extends only 35 to 45 cm out from the end wall. At later times, flame lengths under the B8811 ceiling are about the same as for the inert ceiling (see Figure 3).

2.2.3.4 Aircraft Material No. 224. Hot combustion products from the PMMA wall fire cause tearing and separation of the perforated face of this material from the underlying honeycomb. By 18 minutes after ignition, when wall fire flames are just impinging on the ceiling, the perforated face is torn across the full

width of the channel for a distance of 61 cm from the end wall and blisters in the face material appear for a distance of 116 cm. Separation of the face material from the honeycomb occurs over the entire, 2.4 m ceiling length by about 24 minutes after ignition, when flame length is only some 30 cm. However, flame spread due to ceiling combustion does not occur for this ceiling material. Flame lengths under the ceiling are roughly the same as for the other flat aircraft materials or for the inert ceiling (see Figure 3).

2.2.3.5 PMMA Ceiling. The flame spread process under a PMMA ceiling, with the usual, PMMA wall fire initiation, is described by measured flame lengths provided in Table A-9 as a function of time after ignition. It is evident from Table A-9 that about 18.4 minutes after ignition, self-sustained flame spread begins to occur over the PMMA ceiling surface. Flame lengths then grow almost linearly with time (see Figure 3) at a rate of about 1.95 cm/s (regression coefficient of 0.92) until flame reaches the outlet-end of the channel less than 20 minutes after ignition. During this spread process, a leading portion of flame appears to be quite thin, with detached, weakly luminous flamelets well downstream from this thin flame region. Upstream of the thin flame region, cellular combustion takes place, resulting in a much thicker flame zone. This cellular flame reaches the outlet-end of the channel 7 to 12 s after the arrival of the leading portion of flame. A cellular flame zone then fills the entire ceiling channel. Spillage of some flame from below the 0.23 m deep side-walls of the channel necessitates fire extinguishment about 21 minutes after ignition.

It was found that during the 164 s period from the onset of ceiling combustion (as evidenced by increased flame length compared to the inert case) until fire extinguishment, the ceiling pyrolysis zone propagated a distance of only 115 cm from the end wall. The average pyrolysis zone spread rate during this interval was thus about 0.7 cm/s, compared to a flame spread rate of 1.95 cm/s. Clearly, there was significant flame extension beyond the pyrolysis zone for this case of ceiling fire spread.

The problem of fire spillage from the ceiling channel was solved for a final experiment with a steadily burning PMMA ceiling by the addition of a 0.152 m ceramic paper extension to the 0.23 m deep ceiling side-walls, as shown in Figure 2. During this experiment, the cellular flame zone was observed to be about 76 mm above the bottom edge of the ceramic paper, which coincided with the extent of ceramic paper blackening seen after the fire. Total flame zone thickness during the steady burning process was, thus, about 0.305 m.

2.2.4 TEMPERATURE AND RADIATION MEASUREMENTS. Data on ceiling and gas temperatures, thermal radiation and heat flux obtained during the full-scale tests appear in Tables A-10 through A-16. Measurements are provided as a function of time from ignition for thermocouples (prefix "A"), narrow angle radiometers (prefix "B"), an infrared absorption meter (prefix "C") and wide angle heat flux gages (prefix "D"). In Table A-16, the time-averaged values of instrument outputs are given for the period of steady burning. Further details on instrument locations appear in Table 2. Instruments which malfunction during an experiment are shown with invalid outputs deleted.

Examination of Table A-11 shows that the peak heat flux to the inert ceiling at a horizontal distance of 1.82 m from the PMMA exposure fire is about 2 W/cm² a little more than 43 minutes after ignition. This flux is reduced to about 1.2 W/cm² at 30 minutes after ignition. As shown in Table A-10, the ceiling heat flux gage (D4) is not initially zeroed for the first test but the corrected measured flux to the ceiling 30 minutes after ignition is in good agreement with that from the subsequent test.

#### 2.3 EXPERIMENTAL RESULTS: MODEL TESTS

Results from the elevated pressure experiments are presented in Figures 4 through 18. These figures contain correlations of data on a given material for all pressures, including one atmosphere when relevant. Corrections for the test pressure are made in accordance with pressure modeling requirements<sup>1</sup>, so that good correlation of data is an indication of modeling success. It should be noted, however, that the time origin for many elevated pressure experiments has been shifted to yield the best possible data correlation during the initial stages of ceiling fire spread.

2.3.1 CEILING FIRE SPREAD. Because all characteristic lengths are reduced as  $(p/p_o)^{-2/3}$  and all characteristic solid phase burning times are reduced as  $(p/p_o)^{-4/3}$  in the pressure modeling scheme,  $^{1}$ ,  $^{12}$  the quantities  $\mathbf{x}_f(p/p_o)^{2/3}$  and  $t(p/p_o)^{4/3}$  should be independent of air pressure and result in correlation of data on ceiling flame length,  $\mathbf{x}_f$ , at any absolute pressure, p (where  $p_o$  is the ambient atmospheric pressure) as a function of time, t, after ignition. Figures 4 through 8 demonstrate this correlation technique for the five different ceiling materials. In Figures 4 and 8, the results from the full-scale experiments are seen to be in reasonable agreement with model results for the lowest air pressures of 11.2 and 21.4 atmospheres. A clear divergence from full-scale behavior is seen to occur, however, when the highest air pressure of 31.6 atmospheres is utilized in the model experiments.

The maximum value of  $x_f(p/p_0)^{2/3}$  for complete fire spread to the end of the ceiling is 2.4 m, the length of the full-scale channel. This value is attained for all the elevated pressure, model experiments with PMMA as well as with the three aircraft materials. However, measurements of  $x_f(p/p_0)^{2/3}$  are not always available at elevated pressure, due to obstruction of the camera mirrored field of view by a radiometer. This radiometer interference begins at  $x_f(p/p_0)^{2/3}$  equal to about 1.8 m. In Figure 8, measurements of flame tip position at the end of the channel are shown for tests at 31.6 atmospheres. These data correspond to direct camera views of the beginning of flame outflow from the channel structure.

Use of the pressure modeling correction for the three aircraft materials, as shown in Figures 5, 6 and 7, results in a good correlation of model results for materials No. 223 and 234 but only a rough correlation for the B8811 material. It should be noted that complete flame spread over the full-scale aircraft ceilings does not occur. The complete flame spread over the model ceilings is

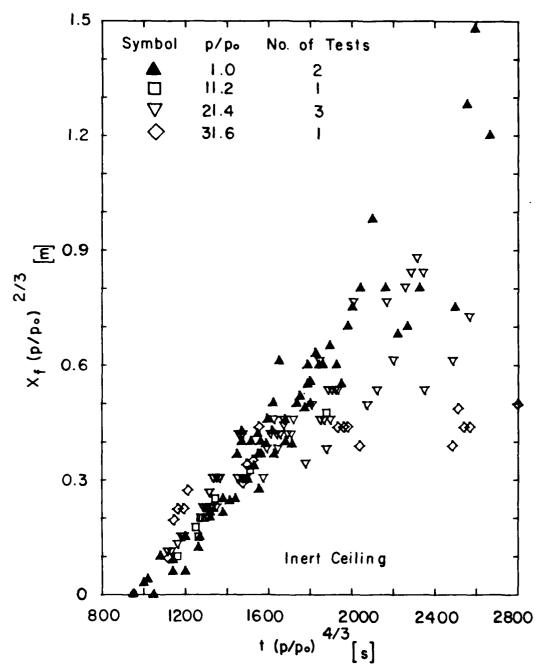


FIGURE 4 CORRELATION OF FLAME LENGTH UNDER INERT CEILINGS

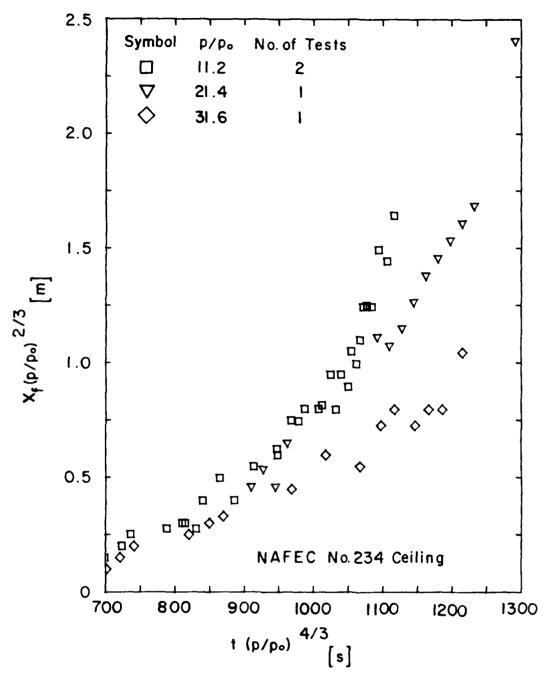


FIGURE 5 CORRELATION OF FLAME LENGTH UNDER NO. 234 MODEL CEILINGS

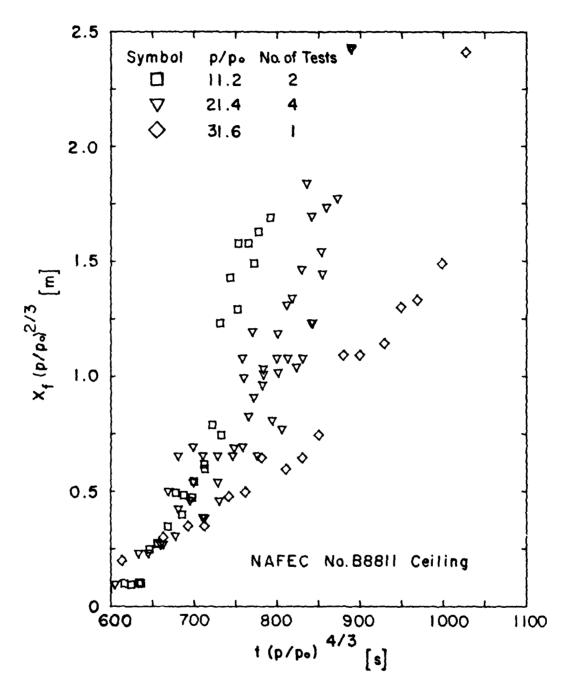


FIGURE 6 CORRELATION OF FLAME LENGTH UNDER NO. B8811 MODEL CEILINGS

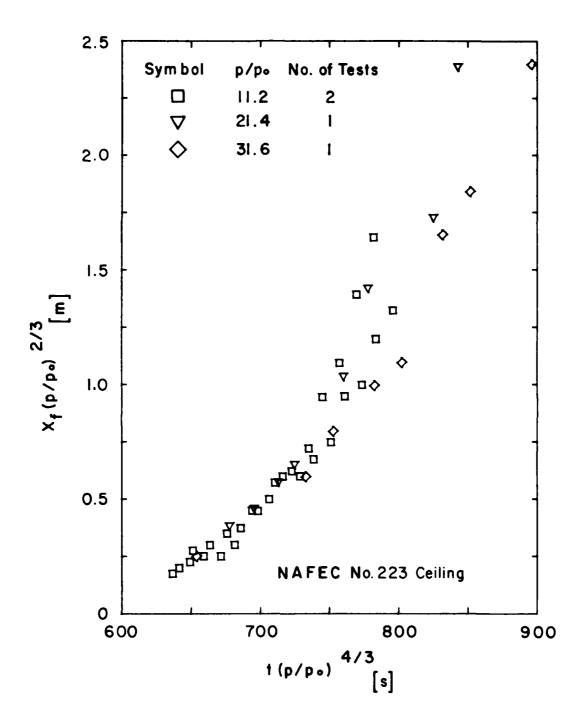


FIGURE 7 CORRELATION OF FLAME LENGTH UNDER NO. 223 MODEL CEILINGS

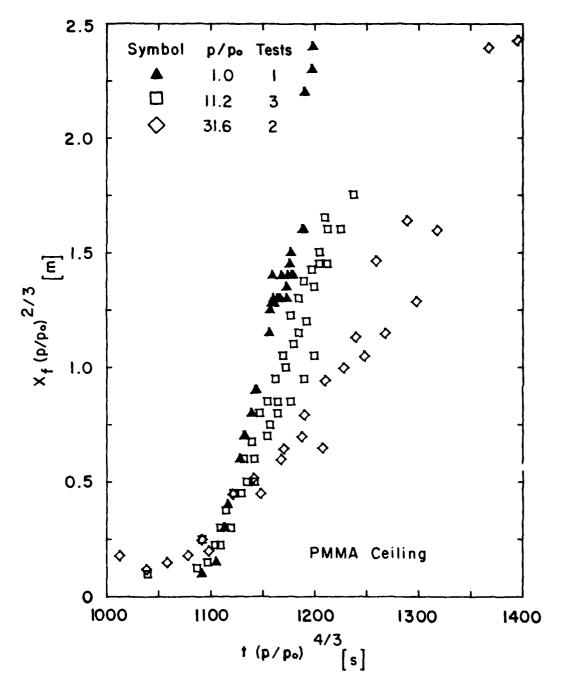


FIGURE 8 CORRELATION OF FLAME LENGTH UNDER PMMA CEILINGS

enhanced by the unimportance of radiative heat loss from hot char layers at elevated pressure and by the use of full-scale ceiling thickness in the models, which provides more fuel than required by the modeling scheme.

For aircraft material No. 223, complete flame spread at elevated pressure occurs only after the perforated surface layer melts and cags, allowing the underlying honeycomb to be exposed.

2.3.2 CEILING FIRE RADIANCE. As explained in references 1 and 12, the pressure modeling scheme results in the preservation of the product of convective heat flux and length-scale (proportional to Nusselt Number since gas temperatures are preserved). With all length scales proportional to  $p^{-2/3}$ , this means that the product,  $p^{-2/3}$ , times convective heat flux, should be independent of air pressure. All heat fluxes, whether convective or radiative, must be consistent with this requirement to insure complete modeling success. Since solid angles and geometric view factors are preserved when all length scales are reduced as  $p^{-2/3}$ , the radiance, N, of ceiling layer gas and ceiling surface should exhibit the same behavior as convective flux. As a result,  $N(p/p_0)^{-2/3}$  should be independent of air pressure.

The pressure corrections derived above for radiance, N, as a function of time, t, from ignition are applied in Figures 9 through 13 to measurements for the five ceiling materials. Agreement with full-scale results is satisfactory for tests at 11.2 atmospheres with inert and PMMA ceilings. At higher ambient pressures for these two ceiling materials, there is clearly not good correlation with transient, one-atmosphere data.

Radiance measurements from model tests with inert ceilings at pressures of 21.4 and 31.6 atmospheres diverge from the one-atmosphere and 11.2-atmosphere results as the ceiling surface temperature increases with increasing test time. The radiance of the hot ceiling, which is nearly independent of pressure, dominates the radiometer output in the absence of flame or dense smoke. Pressure modeling of this ceiling surface radiance is not possible since  $N(p/p_0)^{-2/3}$ , not N, should be independent of pressure.

Table A-16 shows that the steady-state value of ceiling layer radiance for a full-scale test with a PMMA ceiling is 6.43 kW/m²sr. Comparison of this full-scale measurement with the model results in Figure 13 indicates excellent agreement for test pressures of 11.2 and 21.4 atmospheres. For the steadily burning PMMA ceiling, success in modeling ceiling layer radiance is probably due to the dominant effect of flame radiation over ceiling surface radiation, although even flame radiation will cease to be modeled at a sufficiently high pressure. The measured radiance at 31.6 atmospheres does not, in fact, model the full-scale value, apparently due to radiant saturation of the flame, since the peak radiance at this pressure is identical to that at 21.4 atmospheres. A similar behavior characteristic of radiant saturation is exhibited by all the aircraft materials at 31.6 atmospheres (see Table 4 in Sec. 2.5).

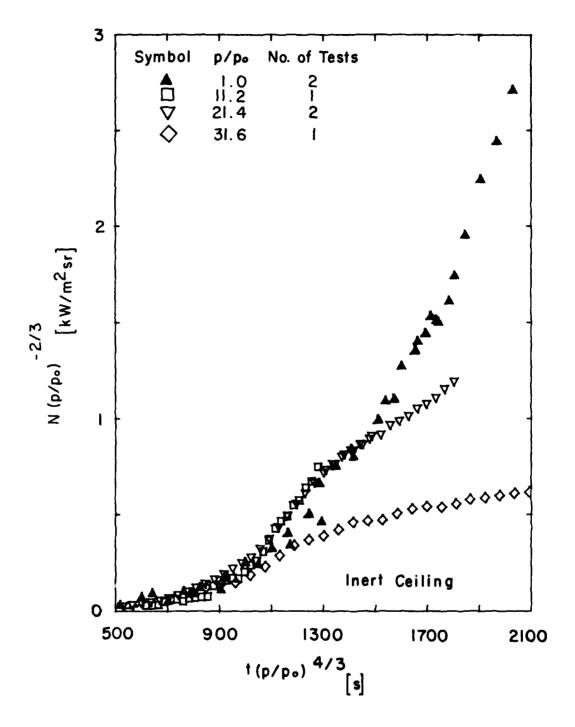


FIGURE 9 CORRELATION OF FLAT AND INERT CEILING RADIANCE

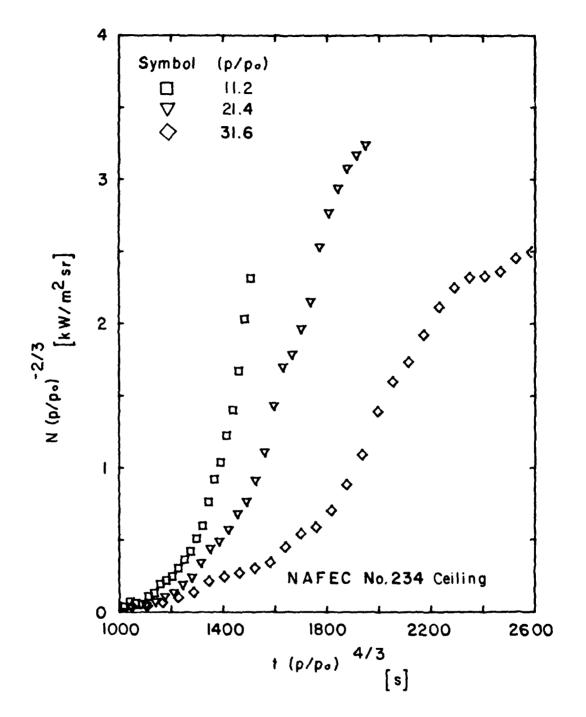


FIGURE 10 CORRELATION OF FLAME AND NO. 234 MODEL CEILING RADIANCE

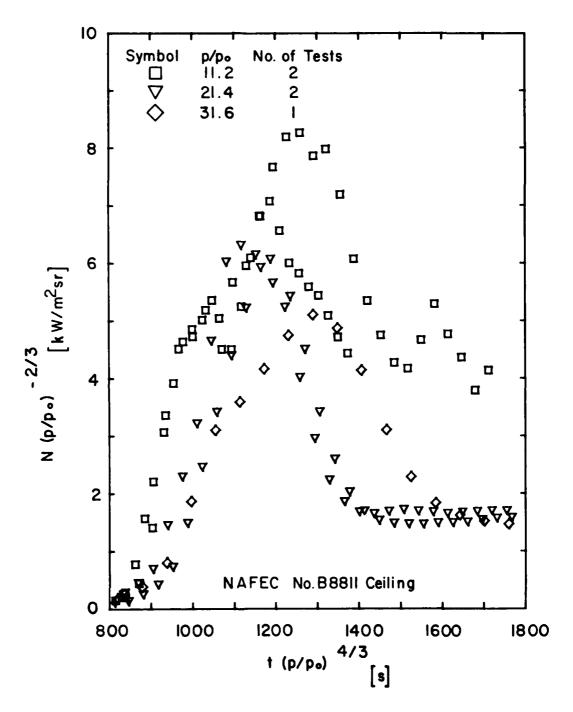


FIGURE 11 CORRELATION OF FLAME AND NO. B8811 MODEL CEILING RADIANCE

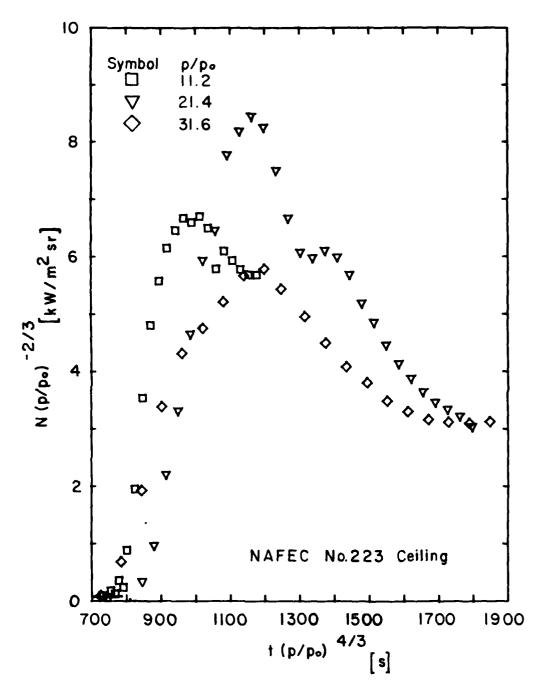


FIGURE 12 CORRELATION OF FLAME AND NO. 223 MODEL CEILING RADIANCE

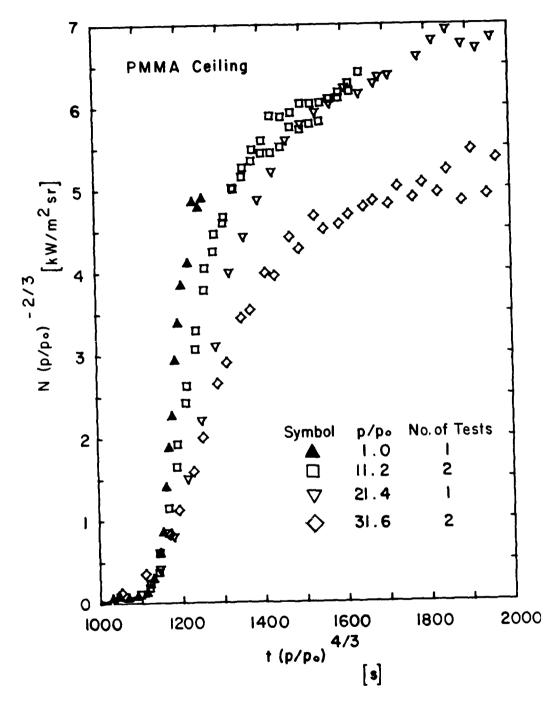


FIGURE 13 CORRELATION OF FLAME AND PMMA CEILING RADIANCE

Correlation of radiance measurements for the three aircraft materials is affected by three factors: 1) the lack of modeling of the flame spread process; 2) the use of a constant material thickness, which leads to an improperly modeled total fire duration; and 3) the limited radiometer response time of about 1.5-2 s. The last factor, which is also relevant for the PMMA ceiling, means that the increase in radiant output cannot be followed accurately during flame front transit at elevated pressure across the roughly 1.3 cm sensing area of the radiometer. Flame transit times at elevated pressures will typically be from 0.1 to 1.0 s.

2.3.3 WALL-CEILING BURNING RATE. By replacing the Nusselt Number with the Sherwood Number, it can be shown  $^{12}$ ,  $^{14}$  that the product of fuel mass flux and a characteristic length scale should be preserved by the pressure modeling scheme. With all length scales proportional to  $p^{-2/3}$ , the fuel mass flux will be proportional to  $p^{4/3}$  times the total mass loss rate,  $\dot{m}$ . The product of fuel mass flux and length scale is then proportional to  $\dot{m}$   $p^{2/3}$ , which should be independent of air pressure.

The pressure-corrected mass-loss rate,  $\dot{\rm m}(\rm p/p_0)^{2/3}$ , is shown in Figures 14 through 18 as a function of pressure-corrected time from ignition. Tests at all three elevated ambient pressures with each of the five ceiling materials are included. In these figures,  $\dot{\rm m}$  represents the mass loss rate of both the channel end-wall and the ceiling, as obtained from a fourth order polynomial regression to the load-cell readings.

The mass loss rate (equal to burning rate in the absence of soot production) shown in Figure 14 corresponds to a measurement for the burning PMMA wall alone, since the ceiling is inert. Correlation of the mass loss rate data in Figure 14 leads to prediction of a steady, full-scale wall burning rate of 4-6 g/s at 1200-1600 s after ignition. Measured wall burning rates, given in Table A-3, of 1.44 to 2.55 g/s are far less than the above steady values predicted by modeling. However, a steady-state condition is not achieved until 2640 s at full scale and measurements correspond to only about 1/3 to 1/2 of this test time. At equivalent times in the model, full-scale burning rates are predicted to be 2 to 4 g/s.

Good correlation of data is obtained for the case of a PMMA ceiling in Figure 18. As noted previously, use of a constant thickness for the aircraft materials results in improper modeling of total fuel quantities, which affects the modeling of peak burning rates. For this reason, burning rates and fire durations at the highest ambient pressures are greater than expected, as shown in Figures 15-17.

# 2.4 ANALYSIS OF MODELING RESULTS

The significance of the preceding data correlations can be better understood by analyzing the dependence of flame heat transfer rates on ambient air pressure. Such an analysis will clarify the problems involved in pressure modeling ceiling flame spread and ceiling fire radiant heat loss.

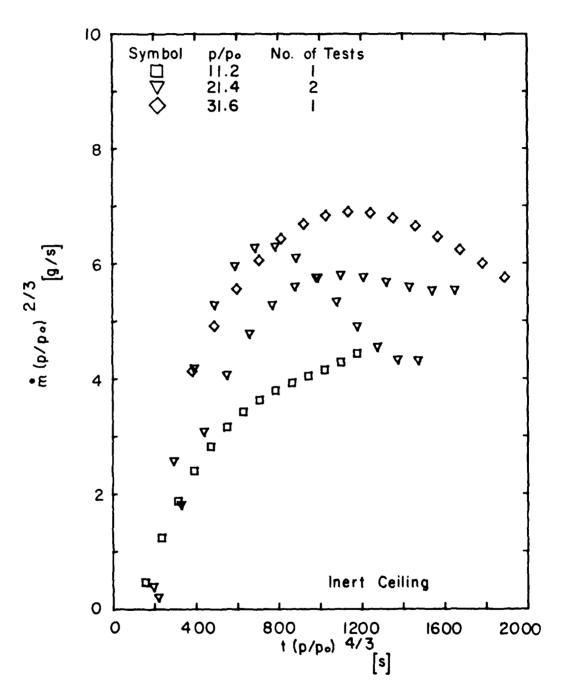


FIGURE 14 CORRELATION OF MASS LOSS PATE FOR INERT MODEL CEILINGS

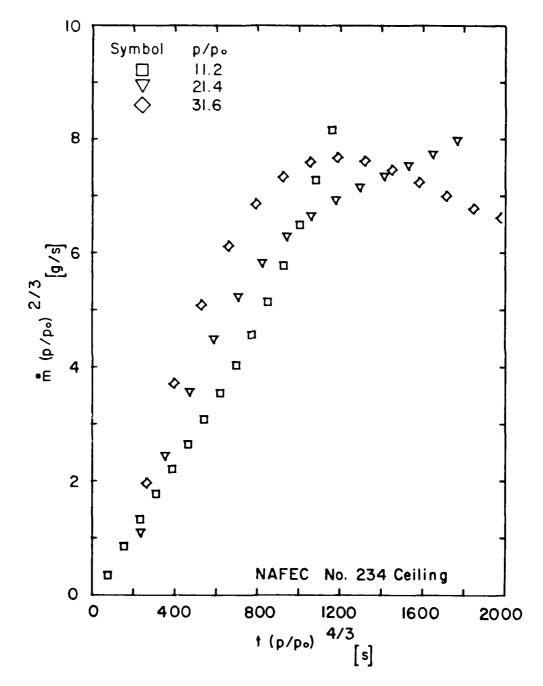


FIGURE 15 CORRELATION OF MASS LOSS RATE FOR NO. 234 MODEL CEILINGS

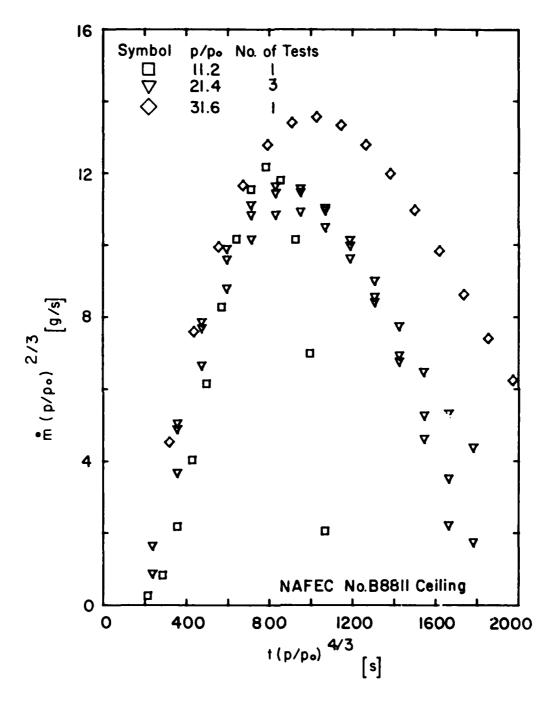


FIGURE 16 CORRELATION OF MASS LOSS RATE FOR NO. B8811 MODEL CEILINGS

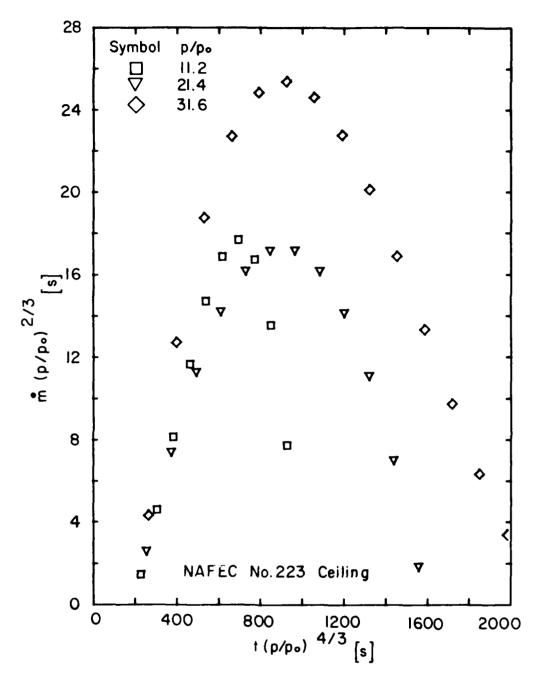


FIGURE 17 CORRELATION OF MASS LOSS PATE FOR NO. 223 MODEL CEILINGS

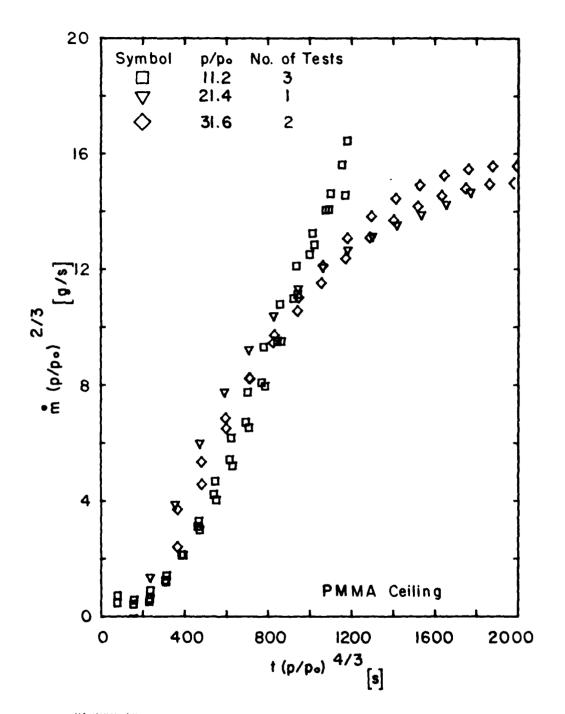


FIGURE 18 CORRELATION OF MASS LOSS RATE FOR PMMA MODEL CEILINGS

2.4.1 HEAT FLUX TO CEILING. The net heat flux,  $\dot{q}$ ", to an inert or fuel ceiling is given by the following expression, previously derived by Orloff et al<sup>15</sup> for the case of a steadily burning wall fire and used by Alpert<sup>1</sup> for the case of upward fire spread:

$$\dot{q}'' = \alpha T_g^4 \left[1 - \exp(-kL_m')\right] - \alpha \sigma T_s^4 + \dot{q}_c''$$
 (1)

where  $L_m^+$  is the actual mean beam length at absolute pressure, p, and the remaining symbols are defined in the nomenclature. The constant,  $\alpha$ , is taken to be 1.0 for steady burning and 0.5 for a spreading fire when  $T_S$  is the pyrolysis temperature of the ceiling material. During fire spread across the channel ceiling, the convective heat flux,  $\dot{q}_C^+$ , as well as the flame radiative properties are influenced by external factors such as the imposed flow from the PMMA wall fire, the conditions at the channel exit and by thermal radiation from the channel side-walls. These factors will be ignored here to simplify the analysis. Instead, conditions typical of PMMA wall fires will be assumed in order to obtain approximations for  $T_p$ , k and  $\dot{q}_C^+$  during the ceiling fire.

As shown in reference 17, gas temperature,  $T_g$ , is found to be preserved at elevated pressure while absorption coefficient, k, is found to increase in wall fires as the 4/3 power of pressure. An increase in k at least proportional to pressure is expected if the ratio of fuel mass fraction converted to soot to the density of a soot particle is independent of pressure. Small increases in soot fraction with pressure could then give the result,  $k=k_0(p/p_0)^{4/3}$ . The convective heat flux in PMMA wall fires is found 15 to be about 5.5 kW/m² at one atmosphere. It is assumed here that this flux is accurately pressure modeled, leading to the quantity,  $\dot{q}_0^{\mu} = \frac{1}{2} \frac{1}{3}$ , being independent of ambient pressure. As a result,  $\dot{q}_0^{\mu} \approx 5.5 \; (p/p_0)^{2/3} \; kW/m²$ .

The radiation mean beam length,  $L_{\rm m}^{\prime}$ , in the ceiling channel, will be reduced as  $p^{-2/3}$ , together with all other length scales. This quantity can, therefore, be expressed as  $L_{\rm m}^{\prime} = L_{\rm m} \; (p/p_{\rm O})^{-2/3}$ , where  $L_{\rm m}$  is the mean beam length at one atmosphere.

With the preceding assumptions used in equation (1), the heat flux to the channel ceiling becomes:

$$\dot{q}'' = \sigma T_g^4 \left[1 - \exp\left(-k_o L_m(p/p_o)^{2/3}\right)\right] - \alpha \sigma T_g^4 + 5.5 \left(p/p_o\right)^{2/3}$$
 (2)

2.4.2 FLAME SPREAD. The rate of flame spread under a combustible ceiling should be inversely proportional to the time,  $\Delta t$ , required to heat the ceiling to its pyrolysis temperature,  $T_{\rm S}$ , from an initial temperature of  $T_{\rm co}$ . For a thermally thick fuel,

$$\Delta t \approx \rho c \lambda (T_s - T_{\omega})^2 / \dot{q}^{n^2}$$
 (3)

where  $\rho C\lambda$  is the product of fuel density, specific heat and thermal conductivity and where  $\dot{q}^{n2}$  is given by equation (1).

Thus, spread rates across thermally thick fuels are directly proportional to the square of the net heat flux to the fuel. The pressure dependence of this quantity is examined in Figure 19. In this figure,  $\dot{q}''^2(p/p_0)^{-4/3}$  should be independent of ambient air pressure, according to the modeling scheme, and equal to the value,  $\dot{q}''^2$ , at one atmosphere, or full-scale. Clearly, such is not always the case. The variation of pressure-corrected heat flux is especially large for k<sub>0</sub> greater than about 0.17.

An estimate of the mean beam length for a fire in the ceiling channel can be obtained from reference 18, where it is shown that

$$L_{\rm m} \approx 1.75 \text{ Wd/(W+d)} \quad . \tag{4}$$

In equation (4), W is the width of the flame volume (the channel width of 0.46 m) and d is the flame depth (at most the 0.23 m depth of the channel side-walls). The maximum value of  $L_m$  for a ceiling channel filled with flame is then, from equation (4),  $L_m = 0.267 \ m$ . If the flame absorption coefficient is typical of PMMA, then  $k_0 = 1.3$  to  $1.5 \ m^{-1}$  from references 19 and 20. The resultant value for  $k_0 L_m$ , about 0.4, is seen in Figure 19 to lead to significant errors in pressure modeling in the 20- to 30-atmosphere range, since the squared net heat flux to the ceiling after pressure correction will be only about 20% of that at one atmosphere. On the other hand,  $k_0 L_m$  at a 1 m elevation on a 0.3 m wide PMMA wall fire (flame thickness about 0.07 m) is only 0.13 to 0.15, leading to a pressure-corrected, squared net heat flux to the wall nearly 80% of the full-scale value. Pressure modeling of fire spread should, thus, be more successful for 1 m high walls than for ceilings in a long, channel configuration. This explains why correlation of fire spread data for a PMMA ceiling in Figure 8 is not as good as the correlations for PMMA wall fire spread in reference 1.

Figure 19 also shows that the occurrence of fuel charring, which tends to elevate surface temperatures, can result in squared heat fluxes to the fuel at 10 to 30 atmospheres which are much greater than those at one atmosphere, even after the appropriate pressure corrections are applied. These increased heat fluxes can explain why complete fire spread occurs for all aircraft ceilings at elevated pressures but not at one atmosphere. Such behavior is basically due to the fact that radiant heat loss from fuel surfaces at elevated pressure (but not at one atmosphere) is generally insignificant compared to flame radiative feedback to the fuel. This same insignificance of solid surface radiant heat loss at elevated pressures is probably responsible for the lack of modeling of flame lengths under the inert ceiling, as shown in Figure 4. It is radiant heating of the PMMA wall fire by the hot ceiling which causes the increases in flame length under the ceiling.

2.4.2 RADIANCE. The radiance, N, of both the hot gases in the ceiling channel and the ceiling surface itself along a ray normal to the ceiling is given by

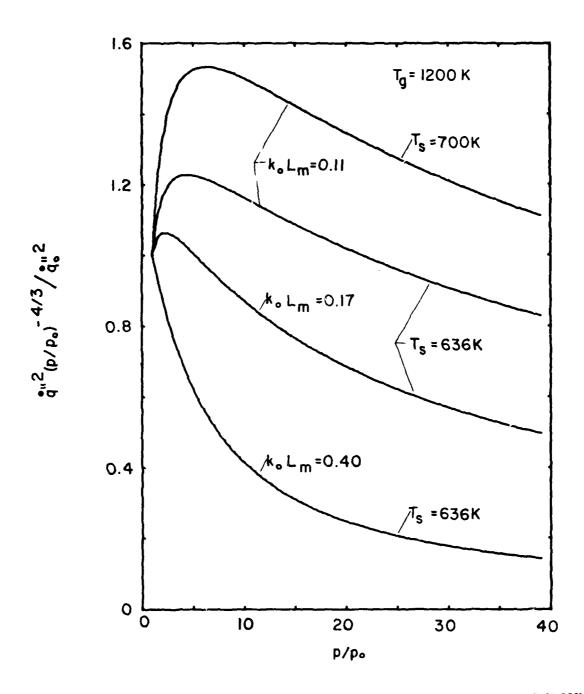


FIGURE 19 PREDICTED BEHAVIOR OF SQUARE OF NET HEAT FLUX TO THERMALLY THICK FUEL SURFACE.

the following expression:

$$N = \frac{\sigma T^{4}}{\pi} \left[ 1 - (1 - T_{s}^{4} / T_{g}^{4}) \exp(-kd') \right]$$
 (5)

where d' is the actual depth of ceiling layer gases at any absolute pressure, p. Equation (5) is applicable to wall lires as well as ceiling fires. In terms of quantities at one atmosphere, or full-scale, equation (5) becomes:

$$N = \frac{\sigma T^{4}}{\pi} \left[ 1 - \left( 1 - T_{s}^{4} / T_{g}^{4} \right) \exp(-k_{o} d(p/p_{o})^{2/3}) \right] . \tag{6}$$

As noted previously, the quantity,  $N(p/p_0)^{-2/3}$ , should be independent of air pressure in the modeling scheme. The plots of ceiling layer radiance in Figure 20, which are obtained from equation (6), show that this modeling condition is not satisfied, even after the pressure correction,  $(p/p_0)^{-2/3}$  is applied. In this figure,  $k_0d = 0.076$  is typical of a 5 cm layer thickness or a 0.81 m elevation on a PMMA wall fire, while  $k_0d = 0.34$  is the maximum value expected for a PMMA ceiling above a 23 cm deep channel filled with flame.

Clearly, the radiance of either a burning wall or a burning ceiling at elevated pressure will be far less than the equivalent value at one atmosphere. On the basis of the plots in Figure 20, ceiling layer radiance measured at 10 to 30 atmospheres will be only 1/2 to 1/3 the respective values predicted by modeling theory for a given, full-scale radiance measurement. Such errors are reduced somewhat, as shown in Figure 20, if the radiance measurement is due mainly to the hot gas layer and not a hot ceiling surface. This effect can be seen in Figure 9, where radiance measurements obtained at elevated pressure diverge from the corresponding, full-scale values as the inert ceiling surface heats up and becomes an important factor in the ceiling layer radiance.

# 2.5 COMPARISON OF CEILING FIRE GROWTH

The data obtained at elevated pressures on flame length, radiance, and fuel mass loss for all five ceiling materials have been matched to both exponential and linear, (least squares) regression fits. Generally, exponential and linear data fits are about equally good, with regression coefficients ranging between 0.88 and 0.96.

Table 3 lists the exponential growth factors resulting from regression fits to the flame length, fuel mass loss and ceiling layer radiance. A fit for the data on  $\mathbf{x}_f$  proportional to  $\mathbf{e}^{\gamma t}$  implies a flame spread velocity,  $V_f$ , given by

$$V_{f} = \frac{dx_{f}}{dt} = \gamma x_{f}$$

The exponential growth factor,  $\gamma$ , is thus the ratio,  $V_f/x_f$ . Similarly, exponential fits to the mass loss,  $\Delta m$ , and radiance data yield  $\dot{m}/\Delta m$  and  $\dot{N}/N$ .

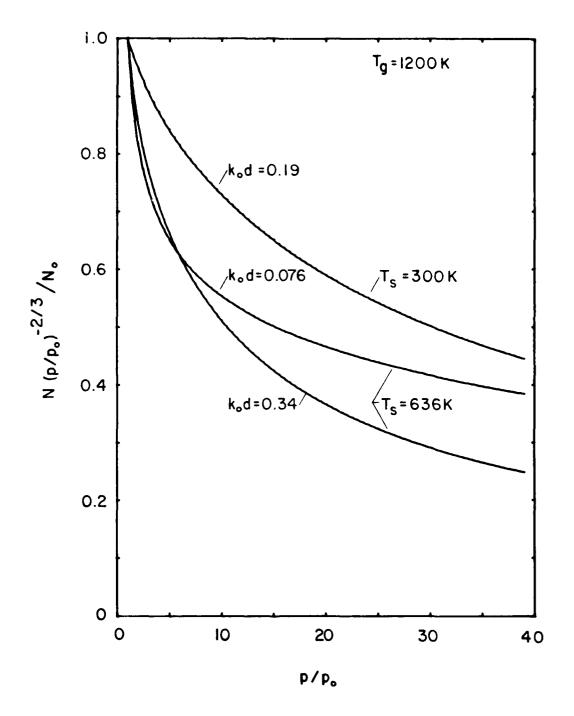


FIGURE 20 PREDICTED BEHAVIOR OF FLAME AND FUEL SURFACE RADIANCE

TABLE 3.-CEILING FIRE EXPONENTIAL GROWTH FACTORS

Ceiling	Flame Spread		Mass Loss	
Material	$v_f/x_f [s^{-1}]$		m⁄∆m [s <sup>-1</sup> ]	ท้/ท [s <sup>-1</sup> ]
		$p/p_0 = 11.2$		
Inert	0.046	· ·	0.056	0.12
РММА	0.40		0.079	0.36
No. B8811	0.44		0.082	0.53
No. 223	0.34		0.10	0.74
No. 234	0.13		0.057	0.195
		$p/p_0 = 21.4$		
Inert	0.073	Ū	0.0745	0.25
P <b>MMA</b>	N.A.		0.099	0.66
No. B8811	0.54		0.12	0.75
No. 223	0.645		0.15	0.85
No. 234	0.245		0.082	0.31
		$p/p_0 = 31.6$		
Inert	0.032	v	0.062	0.32
P <b>MMA</b>	0.82		0.10	1.13
No. B8811	0.52		0.14	0.86
No. 223	0.92		0.22	1.77
No. 234	0.38		0.071	0.50

N.A. = not available due to camera malfunction

Table 3 shows that aircraft material No. 234 behaves similarly to the inert material as far as mass loss and radiance are concerned but has a flame spread rate about three times that of the inert material. The remaining three materials, including PMMA, have much larger growth factors than material No. 234 at all elevated pressures. Of the three materials with the higher growth factors, values of  $\gamma$  are nearly always largest for aircraft material No. 223 with respect to flame spread, mass loss, and radiance.

Peak values of mass loss rate and radiance are listed in Table 4 for the elevated pressure experiments. Peak flame lengths are not listed since flame spread is complete to the end of the channel for all except the inert material. As with the growth factors in Table 3, the materials can be roughly ranked in the following order of increasing peak mass loss rate or peak radiance at all elevated pressures:

- 1. Inert
- 2. No. 234
- 3. No. B8811
- 4. PMMA
- 5. No. 223

with the last three materials exhibiting very similar peak mass loss and radiance behavior.

TABLE 4.-MAXIMUM MASS LOSS RATE AND RADIANCE OF CEILING FIRES

Ceiling	Peak Mass Loss Rate		Peak Radiance
Material	max [g/s]		N <sub>max</sub> [W/cm <sup>2</sup> sr]
	<del></del>	$p/p_0 = 11.2$	
Inert	>0.9		>0.37
P <b>MM</b> A	>3.2		3.0
No. B8811	2.5		4.1
No. 223	3.5		3.3
No. 234	>1.6		>1.15
		$p/p_0 = 21.4$	
Inert	0.77	-	>0.69
PMMA	1.9		5.5
No. B8811	1.5		4.8
No. 223	2.25		6.5
No. 234	1.0		2.5
		$p/p_0 = 31.6$	
Inert	0.71	-	0.64
PMMA	1.55		5.5
No. B8811	1.4		5.1
No. 223	2.55		5.75
No. 234	0.77		2.56

<sup>&</sup>gt; = experiment time insufficient to determine maximum value

Flows of flaming gases or combustion products in a long ceiling channel, such as that in an aircraft cabin, can undergo significant temperature variations with distance from one end of the ceiling to the other. Such variations in gas temperature along the length of the channel can be due to radiant heat loss to the cooler floor area or to conduction losses to the ceiling surface. If there are ventilation openings in the channel configuration permitting hot gas outflow and cool air inflow, mixing of outflow with inflow may occur, leading to reduced gas temperatures near the opening. On the other hand, combustible ceilings or wall fire impingement on the ceiling will lead to elevated gas temperatures.

A mathematical formulation is developed here to calculate thermal radiation from ceiling layers with temperatures and absorption coefficients nonuniform in three dimensions. This is a generalization of the methods described for uniform ceiling layers  $^{21}$  and for ceiling layers with properties varying in the vertical direction only  $^{22}$ . Some simplified methods of calculating the problem are then tried and compared with the exact numerical procedure to evaluate the accuracy of various approximations.

# 3.1 MATHEMATICAL FORMULATION

Temperatures and absorption coefficients in the ceiling layer are assumed to be known functions of the three spatial coordinates x, y and z:

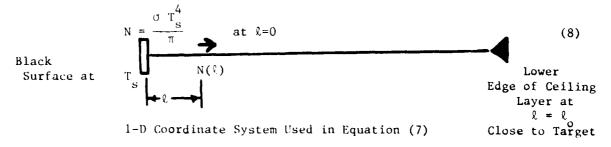
$$T = T(x,y,z)$$

$$k = k(x,y,z) .$$

The one dimensional equation of radiative transfer for the radiance, N, (power per unit area per unit solid angle) along a ray of path length,  $\ell$ , in the ceiling layer which is in thermodynamic equilibrium is,

$$\frac{\mathrm{d}}{\mathrm{d}\ell} \frac{\mathrm{N}(\ell)}{\mathrm{d}\ell} = \frac{\mathrm{k}(\ell) \ \sigma \ \mathrm{T}(\ell)^4}{\pi} - \mathrm{k}(\ell) \ \mathrm{N}(\ell) \tag{7}$$

where  $\sigma$  is the Stefan-Boltzmann constant. For rays viewing the enclosure surfaces bounding the ceiling layer, the boundary condition is:



Integrating equation (7) using (8) over the path length  $\ell_o$  through the ceiling layer yields

$$N(\ell_0) = \exp\left[-\int_0^k k(\ell)d\ell\right] \left\{ \int_0^k \frac{\sigma T^4(\ell)k(\ell)}{\pi} \exp\left[\int_0^k k(u)du\right]d\ell + \frac{\sigma T_s^4}{\pi} \right\} . (9)$$

This can be re-written as

$$N(\ell_0) = \frac{\sigma}{\pi} \int_{0}^{\ell_0} T^4(\ell)k(\ell) \exp\left[-\int_{\ell}^{\ell_0} k(u)du\right]d\ell + \frac{\sigma T_s^4}{\pi} \exp\left[-\int_{0}^{\ell_0} k(\ell)d\ell\right]. \quad (10)$$

The radiative flux  $\dot{q}$ ", from the ceiling layer and enclosure surfaces to a differential target surface element is given by:

$$\dot{q}'' = \int_{\Omega} \frac{N (\vec{n} \cdot \vec{r}_{o})}{r_{o}} d\Omega$$
 (11)

where  $(\vec{n} \cdot \vec{r}_0) \geq 0$  and  $\vec{n}$  is the unit normal to the target element with direction cosines u, v and w;  $\vec{r}_0$  is the radius vector along the line of sight (ray) to the far edge of the ceiling layer;  $(\vec{n} \cdot \vec{r}_0)$  represent the dot product  $(\geq 0)$  of the two vectors; and  $d\Omega$  is the elemental solid angle subtended at the target element. The integration is performed over all solid angles with  $(\vec{n} \cdot \vec{r}_0) \geq 0$  which includes the ceiling layer.

The path length ( can be written in cylindrical coordinates as

$$= \frac{z_0^{-2}}{\cos \theta} \qquad \text{for all } \phi, \ H_1 \le z \le z_0 \le H_2$$

where  $z_0$  is the vertical coordinate of the ray at the far edge of the ceiling layer;  $H_1$  and  $H_2$  are the vertical coordinates of the lower and upper edges of the ceiling layer respectively;  $\theta$  and  $\phi$  are the polar and azimuthal angles respectively; z is the vertical coordinate at  $\ell$ . The origin z=0 is at the target. It is assumed that the region below the ceiling layer is transparent. To transform equation (11) into cylindrical coordinates we use the following three relationships:

$$0 \le \ell \le \ell_0$$
  $z_0 \ge z \ge H_1$ ,

for any function f;

$$\int_{0}^{\ell_{0}} f(\ell) d\ell = \int_{z_{0}}^{H_{1}} f(z,\theta,\phi) \frac{d\ell}{dz} dz = \int_{z_{0}}^{H_{1}} - \frac{f(z,\theta,\phi)}{\cos\theta} dz = \int_{H_{1}}^{z_{0}} \frac{f(z,\theta,\phi)}{\cos\theta} dz$$
;

and in particular

$$\int_{\ell}^{\ell} k(u) du = \int_{H_{1}}^{z} \frac{k(s, \theta, \phi)}{\cos \phi} ds$$

and also

$$d\Omega = \sin\theta d\phi d\theta$$
.

So the flux received by the target can be written as

$$\dot{q}'' = \frac{\sigma}{\Pi} \int_{\theta=0}^{\pi} \int_{\phi=0}^{\pi} \int_{\phi=0}^{\pi} \frac{(\dot{n} \cdot \dot{r}_{o})}{r_{o}} \sin\theta \int_{z=H_{1}}^{z} \frac{T^{4}(z,\theta,\phi)k(z,\theta,\phi)}{\cos\theta} \exp[-\int_{H_{1}}^{z} \frac{k(s,\theta,\phi)}{\cos\theta} ds]dzd\phid\theta$$

$$+ \frac{\sigma T_{s}^{4}}{\Pi} \int_{\theta=0}^{\pi} \int_{\phi=0}^{\pi} \frac{1}{r_{o}} \frac{(\dot{n} \cdot \dot{r}_{o})}{r_{o}} \exp[-\int_{cos\theta}^{z} \frac{k(s,\theta,\phi)}{\cos\theta} ds]d\phid\theta$$

$$H_{1}$$

where "\*" indicates that  $(n \cdot r_0) \ge 0$ .

Now  $(\overrightarrow{n} \cdot \overrightarrow{r}_{0})/r_{0} = u \sin\theta\cos\phi + v \sin\theta\sin\phi + w \cos\theta$ 

as 
$$\vec{n} = \vec{i} u + \vec{j}v + \vec{k}w$$

$$\vec{r}_0 = r_0(\vec{1} \sin\theta \cos\phi + \vec{j} \sin\theta \sin\phi + \vec{k} \cos\theta)$$
.

In order to simplify the preceding computations, it is convenient to divide the enclosure into box-like sections as in Figures 21A and 21B such that the target is located at a corner of each section. The flux reaching the target will be the sum from all sections and each section flux is computed independently of others. Limits of integration are found for a typical section in the following discussion.

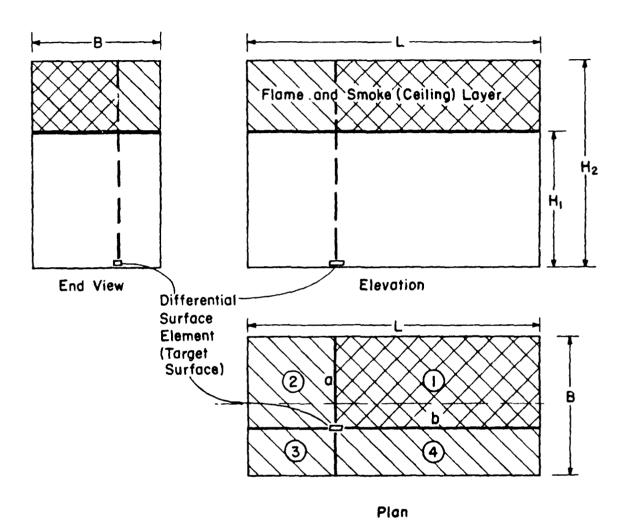


FIG. 21A Elevation, end and plan views of enclosure of length-L x breadth-B x height- $\rm H_2$ . The target is located on the floor of the enclosure. The lower edge of the ceiling layer is at a height  $\rm H_1$  above the target surface. Also shown are the four rectangular parallelepipeds (boxes) used to calculate the radiative flux from the ceiling layer to the target element. The target is at one corner of each rectangular parallelepiped. The ceiling layer in rectangular parallelepiped No. 1 of length-b x breadth-a x height- $\rm H_2$  is shown by the cross-hatched region in each view, (b  $\geq$  a). This figure is from reference 21.

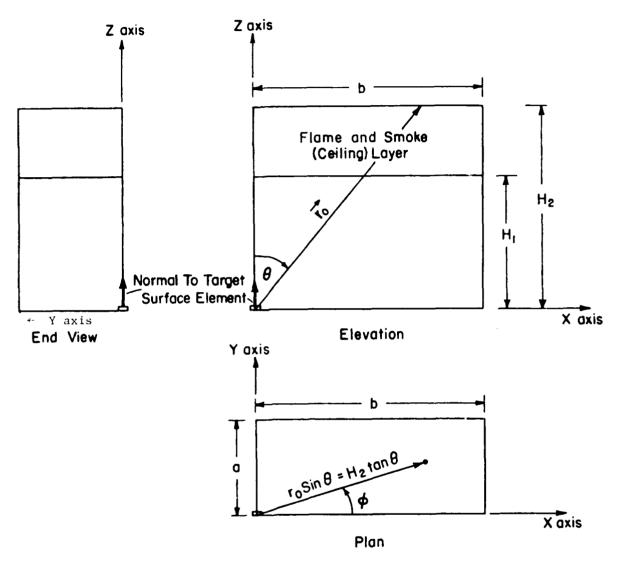


FIG. 21B Detail of elevation, end and plan views of rectangular parallelepiped No. 1 of length-b x breadth-a x height-H2. The lower edge of the ceiling layer is at a height H1 above the target surface. The target is at one corner of the rectangular parallelepiped. In this particular case the target normal is shown to coincide with the z-axis. The origin of a Cartesian and a spherical polar coordinate system is located at the target element.  $\theta$  and  $\phi$  are the polar and azimuthal angles respectively for the spherical polar coordinate system. The rectangular coordinate axes are selected such that  $b \geq a$ . This figure is from reference 21.

It is assumed that the absorption coefficient below the ceiling layer is zero, so that contributions from this region do not influence the results. This fact, together with the target direction and the section boundaries, determine the limits. Since each section flux is computed independently, direction cosines of target normal are to be redefined for each section, according to the section coordinate setup.

The flux from one section is given by:

$$\dot{q}_{1}^{"} = \frac{\sigma}{\pi} \int_{\theta=0}^{\star} \sin\theta \int_{\phi=\phi_{\ell}(\theta)}^{\star} (u \sin\theta \cos\phi + v \sin\theta \sin\phi + w \cos\theta)$$

$$\dot{q}_{1}^{"} = \frac{\sigma}{\pi} \int_{\theta=0}^{\star} \sin\theta \int_{\phi=\phi_{\ell}(\theta)}^{\star} (u \sin\theta \cos\phi + v \sin\theta \sin\phi + w \cos\theta)$$

$$z = H_{1}$$

where "\*" indicates  $\theta$  and  $\phi$  such that (u sin $\theta$ cos $\phi$  + v sin $\theta$ sin $\phi$  + w cos $\theta$ )  $\geq$  0. and where a — is the width of the section

b is the length of the section

 $^{\mathrm{H}}1$  the lower edge of the ceiling layer measured up from the height of the target

 $\mathrm{H}_2$  the upper edge of the ceiling layer or the height of the enclosure measured up from the target

$$\begin{array}{lll} \phi_{\ell}(\theta) = 0 & \phi_{u}(\theta) = \pi/2 & \text{for } 0 \leq \theta \leq \tan^{-1}(a/H_{1}) \\ \\ \phi_{\ell}(\theta) = 0 & \phi_{u}(\theta) = \sin^{-1}(\frac{a}{H_{1}\tan\theta}) & \text{for } \tan^{-1}(\frac{a}{H_{1}}) \leq \theta \leq \tan^{-1}(b/H_{1}) \\ \\ \phi_{\ell}(\theta) = \cos^{-1}(\frac{b}{H_{1}\tan\theta}), \ \phi_{u}(\theta) = \sin^{-1}(\frac{a}{H_{1}\tan\theta}) & \text{for } \tan^{-1}(\frac{b}{H_{1}}) \leq \theta \leq \tan^{-1}(\sqrt{\frac{a^{2}+b^{2}}{H_{1}}}) \\ \\ z_{o} = H_{2} & \text{for } 0 \leq \theta \leq \tan^{-1}(a/H_{2}) \\ \\ z_{o} = \min \min \text{ of } (\frac{b}{\tan\theta\cos\phi}, H_{2}) & \text{for } \tan^{-1}(\frac{a}{H_{2}}) \leq \theta \leq \tan^{-1}(\sqrt{\frac{a^{2}+b^{2}}{H_{1}}}) \\ \\ \phi_{\ell}(\theta) = \cos^{-1}(\frac{b}{H_{1}\tan\theta}) & \text{for } \tan^{-1}(\frac{a}{H_{2}}) \leq \theta \leq \tan^{-1}(\sqrt{\frac{a^{2}+b^{2}}{H_{1}}}) \\ \\ \phi_{\ell}(\theta) = \cos^{-1}(\frac{b}{H_{1}\tan\theta}) & \text{for } \tan^{-1}(\frac{a}{H_{2}}) \leq \theta \leq \tan^{-1}(\sqrt{\frac{a^{2}+b^{2}}{H_{1}}}) \\ \\ \phi_{\ell}(\theta) = \cos^{-1}(\frac{b}{H_{1}\tan\theta}) & \text{for } \tan^{-1}(\frac{a}{H_{2}}) \leq \theta \leq \tan^{-1}(\sqrt{\frac{a^{2}+b^{2}}{H_{1}}}) \\ \\ \phi_{\ell}(\theta) = \cos^{-1}(\frac{a}{H_{2}}) \leq \theta \leq \tan^{-1}(\frac{a}{H_{2}}) \\ \\ \end{array}$$

$$\mathcal{L}_{o} = \min \min \left(\frac{1}{\tan^{-1}(\frac{a}{H_{2}})}\right) \leq \frac{1}{\tan^{-1}(\frac{a}{H_{2}})} \leq \frac{1}{\tan^{-$$

## 3.2 APPROXIMATE SOLUTION

An analytic approximation for the above numerical formulation is available when the enclosure bounding the gas layer is a rectangular parallelepiped, temperature and absorption coefficient are uniform and the target surface is parallel to the ceiling layer, as shown in reference 21. When properties change only in the vertical (z) direction, this analytic technique is modified somewhat by use of an equivalent layer temperature,  $\overline{\Gamma}^4$ , and absorption coefficient, k, defined in reference 22 as follows:

$$\frac{1}{T^4} = 2 \int_{0}^{H_2} T^4(z)k(z)E_2\left(\int_{0}^{z} k(z)dz\right) dz$$

$$\frac{H_2}{1 - eE_3\left(\int_{0}^{z} k(z)dz\right)} \tag{13}$$

$$\hat{\mathbf{k}} = (\hat{\mathbf{T}} - \hat{\mathbf{T}}_{\alpha}) \int_{0}^{\mathbf{H}_{2}} \mathbf{k}(\mathbf{z}) d\mathbf{z} / \int_{0}^{\mathbf{H}_{2}} (\hat{\mathbf{T}}(\mathbf{z}) - \hat{\mathbf{T}}_{\infty}) d\mathbf{z}$$
(14)

where  $\overline{T} = (\overline{T}^4)^{1/4}$  and  $E_2$ ,  $E_3$  are the second and third exponential integrals, respectively.

A layer depth,  $H_2$ -  $H_1$  is defined by:

$$H_2 - H_1 = f \qquad (T(z) - T_{\infty}) dz / (\widetilde{T} - T_{\infty})$$
(15)

The equivalent radiant properties, defined above, can then be used to compute the radiative flux,  $\dot{q}''$ , to a target at the corner of one of the four rectangular parallelepipeds in the enclosure (see Figure 21) by means of the following analytic approximation developed in reference 21:

$$\dot{q}'' = \frac{\sqrt{T^4}}{2\pi} \{1 - (1 - \frac{T^4}{2\pi}) \exp(-\bar{k} L_m)\} V$$
 (16)

where 
$$L_{m} = \frac{2 \text{ ab } (H_{2} - H_{1})}{(H_{2} - H_{1}) (a+b) + ab}$$

and 
$$V = \frac{a}{\sqrt{H_1^2 + a^2}} tan^{-1} \left( \frac{b}{\sqrt{H_1^2 + a^2}} \right) + \frac{b}{\sqrt{H_1^2 + b^2}} tan^{-1} \left( \frac{a}{\sqrt{H_1^2 + b^2}} \right)$$

and layer depth,  $\mathrm{H_2}$ -  $\mathrm{H_1}$  is given by equation (15).

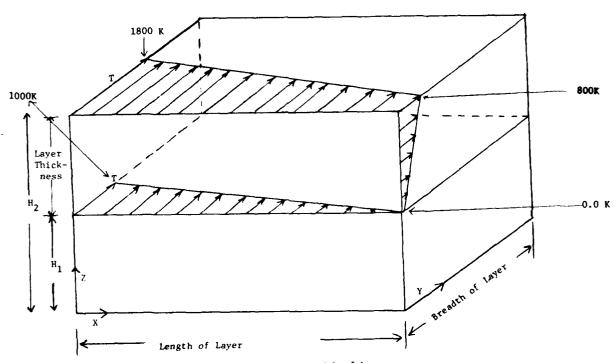
An approximation technique would be valuable in the case of a smoke layer with three-dimensional variations in both temperature and absorption coefficient, since huge amounts of computation time are needed to calculate radiative flux with purely numerical methods. To determine the accuracy of such a simplified calculation procedure, a specific ceiling layer with the largest temperature gradients which might exist (during an aircraft cabin fire) is examined. As shown in Figure 22, the temperature distribution assumed for the gas layer is a linear function with respect to height, z, and length, x, and constant with respect to width, y. Radiation from the ceiling surface is neglected and the absorption coefficient is taken as constant to see effects of gas temperature more clearly.

Results of several calculation procedures with the assumed temperature profile are shown in Table 5 for an upward facing target 1 m below the bottom of a 1 m thick layer at the mid-length point while Table 6 pertains to a target at one end of the layer. In the first row of Tables 5 and 6 are the exact numerical solutions for absorption coefficients of 1.2 m<sup>-1</sup> and 0.5 m<sup>-1</sup>. The second row of Table 5 gives a simplified numerical result obtained by assuming that the vertical distribution of temperature at the target location  $(x_T, y_t)$  is the same distribution in z at all x,y locations in the gas layer. Results in the third and last rows of Table 5 are obtained by the analytic method<sup>21,22</sup> discussed above, with the equivalent temperature,  $T^4$ , derived from the vertical temperature distribution at the target location alone.

For the present example, the analytic method is seen to be very accurate when the target is below the center of the layer. However, Table 6 shows that the analytic method overestimates the radiant flux by 13 to 15% when the target is below one end of a layer. The latter case is a rather stringent test of the analytic approximation since a real gas layer would not normally have an abrupt end, with no further contribution to the target radiant flux from a burning end-wall or outflow region. It is also seen from Tables 5 and 6 that the accuracy of the approximation technique will not be significantly affected by the magnitude of the gas absorption coefficient.

## 3.3 APPLICATION TO FULL-SCALE TEST RESULTS

The preceding techniques can be used to examine the self-consistency of the temperature and heat flux measurements made during the full-scale ceiling channel tests. Measured values of radiance and of radiant flux are compared



Temperature profile is assumed to be uniform along breadth of layer.

FIGURE 22 BASIC TEMPERATURE DISTFIBUTION USED FOR CALCULATED RESULTS IN TABLES 5 AND 6

TABLE 5.-RADIANT FLUX TO TARGET LOCATED BELOW CENTER OF LAYER

Lay	er	Kind of	Temp. Distn.	Kind of	Flux in [W/c	$m^2$ ] for
Dim	ensions	Approximation	T(x,y,z)	Computation	Absorption C	oefficient
	[ m ]		[K]		$k=1.3 [m^{-1}]$	$k=0.5 [m^{-1}]$
1)	30x4x1	None	200-33.33x	Numerical	2.11	1.7
	$H_{1} = 1$		+800z			
2)	11	Local Vertical	-300+800z	Numerical	2.08	1.68
		Distribution				
		for all x,y				
3)	ti	For all $\underline{x}, y, z$ $T^4 = T^4$	844.4	Analytical	2.09	
		$\overline{k} = 1.3$				
		Temperature				
		Distribution				
		of Line 2				
4)	11	$T^4 = T^4$	916.45	Analytical		1.69
		$\overline{k} = 0.5$				
		Temperature				
		Distribution				
		of Line 2				

TABLE 6.-RADIANT FLUX TO TARGET LOCATED BELOW END OF LAYER

Layer		Kind of T	emp. Distn.	Kind of	Flux in [W/cr	m <sup>2</sup> ] for
Dimen	sions	Approximation	T(x,y,z)	Computation	Absorption Co	oefficient
[ m	1]		[K]		$k=1.3 [m^{-1}]$	$k=0.5 [m^{-1}]$
1) 30	x4x1	None	200-33.33x	Numerical	5.44	3.90
Н <sub>1</sub>	= 1		+ 800z			
2)	11	For all $\underline{x}$ , $y$ , $z$ $T^4 = T^4$	1311.93	Analytical	6.13	
		$\overline{k} = 1.3$				
		using local				
		vertical				
		distributio	n			
3)	**	For all $\underline{x}$ , $y$ , $z$ $T^4 = T^4$	1386.7	Analytical		4.49
		k=0.5				
		using local				
		vertical				
		distribution				
4) 1	0x4x1	None	200-100x	Numerical	4.54	3.18
Н	_i= 1		+ 800z			

with values computed from temperature distributions inferred and extrapolated from the experimental data. Three cases are examined: tests with an inert ceiling; a spreading fire under a PMMA ceiling and a steadily burning PMMA ceiling. Data for the inert ceiling are obtained from Table A-11 for a quasi-steady period just before extinguishment at 2700 s. Table A-15 is used for the spreading PPMA fire, at a time after ignition of 1257.6 s (extinguishment about 5 s thereafter), while Table A-16 contains the data on the steadily burning PMMA fire.

3.3.1 DISTRIBUTION OF PROPERTIES. The positions of the thermocouples, absorption coefficient instrument and radiometers are shown in Figure 2. With the data from these instruments and the following assumptions, temperature and absorption coefficient distributions for the whole ceiling layer are found.

Assumption 1. All properties do not vary along the width (y direction); i.e.,  $P(x, y_1, z) = P(x, y_2, z)$  for all  $y_1$  and  $y_2$ .

Assumption 2. Properties vary along the length (x direction) exactly the same way at all heights. Similarly the variation along the height is the same at all lengths;

i.e., 
$$P(x_1, y, z_1) - P(x_1, y, z_2) = P(x_2, y, z_1) - P(x_2, y, z_2)$$
  
for all  $x_1, x_2, z_1, z_2$ .

So, if 
$$P_{x_1}(z) = b_0 + b_1 z + b_2 z^2 + b_3 z^3$$
 at  $x = x_1$ 

and 
$$P_{z_1}(x) = a_0 + a_1 x + a_2 x^2 + a_3 x^3$$
 at  $z = z_1$ ,

a two dimensional distribution satisfying the requirements is

$$P(x,z) = C_0 + a_1 x + a_2 x^2 + a_3 x^3 + b_1 z + b_2 z^2 + b_3 z^3$$

where 
$$C_0 = P* + (b_0 - P_{x_1}(z_1)) + a_0 - P_{z_1}(x_1)$$
.

 $P^* = Actual Property P at (x_1, z_1)$  (ideal or observed value).

Since there is only one set of thermocouples along the height and another set along the length, polynomial fits of the observed temperatures are made along height and along length and combined as above to give a two-dimensional fit. This temperature distribution is evaluated at extreme points to make sure that the range of temperature is realistic.

As there is only one location where an absorption coefficient is measured, information on absorption coefficient is limited. The measured value is ob-

tained very near the ceiling. For both cases where the ceiling is burning PMMA, the absorption coefficient, which is influenced by the PMMA fuel vapor, is found to be  $3.5~{\rm m}^{-1}$ . Since the absorption coefficient for a PMMA flame is known to be about  $1.55~{\rm m}^{-1}$  (from reference 19), an average value of  $2.5~{\rm m}^{-1}$  is used in one calculation. A fit for absorption coefficient over z, varying from 1.5 (where temperature is maximum) to 3.5 (where k is observed) is used in a second calculation. The absorption coefficient is assumed to be constant with respect to x and y.

3.3.2 ENCLOSURE APPROXIMATIONS. As shown in Figure 2, the burning part of the end-wall does not extend to the ceiling and is narrower than the ceiling. However, the calculations are done assuming the whole .46 m wide end-wall is black, and is at 636 K (the PMMA vaporization temperature). There is no wall at the channel outlet. It is assumed that the ceiling is a black surface at a measured temperature or at 636 K (for PMMA) above the layer and that a 300 K temperature exists below the ceiling layer. There are side walls and they are also assumed to be black and at the ceiling temperature inside the ceiling layer and at 300 K (ambient) below it.

When the flux to the  $90^{\circ}$  angle Gardon radiometer is computed, only the 1.2-m length of ceiling which is visible to the radiometer is used. This layer section is taken as a parallelepiped and not as a cone, which is actually seen by the radiometer. The general numerical program can handle the cone structure but the analytic approximation does not.

3.3.3 <u>COMPUTATION RESULTS</u>. Fluxes to the total heat flux gage and the 90° angle radiometer are computed using the numerical scheme described above and the analytical scheme of reference 21. Results are listed in Tables 7 to 9 for the three separate experiments. The computed fluxes are on the whole slightly lower than the measured values, probably due to thermocouple radiation errors giving somewhat lower than actual gas temperatures. On the other hand, computed radiance is much higher than the observed value for the two PMMA ceiling fires, perhaps due to changes in radiometer calibration. Although the measured outputs of both narrow angle radiometers (B1, B2) are generally self-consistent, the slightly higher output of the parallel beam instrument (B2) is used for comparison with the calculations in Tables 7 to 9.

Calculations have also been performed to determine if the measured PMMA wall mass flux is consistent with the radiant flux level from the ceiling surface and gas layer. For a vertical target on the burning PMMA wall surface during the quasisteady period of the second inert ceiling test (Table A-11), calculated radiant fluxes transmitted through the PMMA flame vary from 0.31 W/cm² at 23 cm to 0.8 W/cm² at 69 cm above the ignition point. This radiant feedback should result in a 31% to 62% increase, at these respective locations, in the total heat flux to the fuel (see reference 15) compared to that for a wall burning in the open. Corresponding increases in measured wall mass flux are somewhat lower, 23% to 44% (see Section 2.2.2) since the mass flux measurements represent a time-average over the entire period of fire growth.

TABLE 7. COMPUTED AND OBSERVED HEAT LOSSES, FULL-SCALE TEST 6-19-80, INERT CEILING

	(neation of Target	Size of Area used for	Kind of	Properties	tes	Heat Flux	xni	Radiance	ast.
Target	in Enclosure x,z,[m]	Computation x, y, z, [m]	Computation	T (K)	*_1   _1	[W/cm ] Computed O	T k <sub>1</sub> [W/cm ] [W/cm   sr] [K] [m <sup>-1</sup> ] Computed Observed Computed Observed	[W/cm /sr] Computed Obse	Observed
90° Gardon radiometer 1.6, 0.61	1.6, 0.61	1.2x0.46x0.61	Numerical $T(x,y,z_1)$ 1.5 0.78	$T(x,y,z_1)$	1.5	0.78	0.79	0.56	,
90° Gardon radiometer 1.6, 0.61	. 1.6, 0.61	1.2x0.46x0.61	Analytical	849	1.5	0.74	0.79	ı	1
lotal Gardon gage	1.8, 0.0	2.4x0.46x0.18	Numerical $T(x,y,z_1)$ 1.5	I(x,y,z <sub>1</sub> )	1.5	2.0%	1.99	,	1
Total Gardon gage	1.8, 0.0	2.4x0.46x0.18 Analytical 830	Analytical	830	1.5	1.79*	1.99	•	1
Ray Radiometer	1.8, 0.23	i	Numerical $I(x,y,z_1)$ 1.5	I(x,y,z <sub>1</sub> )	1.5	i	ı	0.54	0.47
T(x,y,z <sub>1</sub> ) = 1.130x10 <sup>3</sup> + 6,378z <sub>1</sub> - 61,341z <sub>1</sub> <sup>2</sup> + 166,020z <sub>1</sub> <sup>3</sup> -202,5 <sup>6</sup> 0z <sub>1</sub> - 578.72x + 350.68x <sup>3</sup> - 89.42x <sup>3</sup> Colling temperature 703 K  Temperature of walls in contact with ceiling layer 703 K Wall temperature below celling layer:  Pref. wall 636 K  All other walls 300K	= 1.130x10 <sup>3</sup> + 6.378z <sub>1</sub> - 61,341z <sub>1</sub> <sup>2</sup> + 166,020z <sub>1</sub> <sup>3</sup> -202,5 <sup>6</sup> 0z <sub>1</sub> <sup>4</sup> - 578.72x + 350.68x <sup>2</sup> - 89.42x <sup>3</sup> - significature 703 K - of walls in contact with ceiling layer 7 <sup>1</sup> - reture below ceiling layer: - if 636 K - walls 300K - of layer .18 m	.020 <mark>23</mark> -202,5402 .42x3 . ceiling layer	1 1 703 K	x is distance from PMMA end-wall [m] y is distance from ceiling side will [m] $z_{\perp}$ is distance from ceiling [m]	nce fr nce fr	om PhMA e om ceilin rom ceili	nd-wall (m g side:wil ng (m)	(E)	

Includes 0.55 W/cm for convective heat flux,

based on a heat transfer coefficient of  $10~\text{M/m}^2k$  (see ref. 20)

TABLE 8. COMPUTED AND OBSERVED RADIANT HEAT LOSSES, FULL-SCALE TEST 8-5-80, PMMA CEILING

	Location of larget	Size of Area used for		Cass Promotive	1	:	:		
1-Mari	بر الد	Computation x, v, z <sub>1</sub> [m]	Kind of Computation	[X]		<pre>Radiant Flux Computed qbserved [W/cm<sup>*</sup>]</pre>	r Flux Qbserved a <sup>-</sup> ]	Rad. Computed [W/cr	Radiance Computed Observed [W/cm <sup>*</sup> /sr]
oo cardon radiometer Wo <sup>o</sup> cardon radiometer	1.6, 0.23, 0.61 1.6, 0.23, 0.61	1.2x0.40x0.01 1.2x0.40x0.61	Numerical Analytical	1(x,y,z <sub>1</sub> ) 997	k(x,y,z <sub>1</sub> ) 2.0	1.27	1.30	68.0	ı
iotal cardon gage lotal cardon gage	1.2, 0.23, 0.76	2.4x0.46x0.76	Numericai Analytical	$\frac{\mathbf{T}(\mathbf{x}, \mathbf{y}, \mathbf{z_1})}{1069}$	κ( <b>κ.γ.ε<sub>1</sub>)</b> 2.5	1.41	1.49	1.17	1
Adv tadiometer	1.8, 0.23, 0.23	ı	Numerical	$\mathbb{I}(\mathbf{x}, \mathbf{y}, \mathbf{z}_1) = \mathbb{K}(\mathbf{x}, \mathbf{y}, \mathbf{z}_1)$	$k(\mathbf{x}, \mathbf{y}, \mathbf{z}_1)$	2.77	1	0.78	0.54
$  x_1,x_2  ^{1-\frac{1}{2}}  \le 971$ $=   y_0     4+1     76     903     z_1   -   1+5     9002     +   988     51                                       $	209.107 x <sup>2</sup> - 7 z <sub>1</sub> - 143,600z <sup>2</sup> 2 z <sub>1</sub> [n <sup>-1</sup> ] 2 z <sub>1</sub> [n <sup>-1</sup> ] 3 ceiling layer	- 71.05506 x <sup>3</sup> 32 <sup>2</sup> + 868,631 z <sup>3</sup> 1 - 1,985,434 z <sup>4</sup> [K] ceiling layer 636 K		s is distant z is distant	x is distance from PRMA end-wall [m] y is distance from ceiling side-wall [m] z <sub>1</sub> is distance from ceiling [m]	Pryk end-wal ng side-wall ing [m]	1 [m] [m]		

Thickness of layer .23 m

TABLE 9. COMPUTED AND OBSERVED RADIANT HEAT LOSSES, FULL-SCALE TEST 8-20-80, PMMA CEILING BURNING STEADILY

	Location	Size of Area		S <b>P</b> ()					
	of larget	used for		Properties	اده	Radian	Radiant Flux	Radiance	nce
	in Enclosure	Computation	Kind of	<b>:-</b>	٠.٤	(computed	Ubserved	Computed	Observed
jažiri	x, y, z [m]	$x, y, z_1$ [m]	Computation	<u>. K</u>	[ m ]	5/M]	[W/cm <sup>2</sup> ]	[W/cm²/sr]	/sr]
90° dardon radiometer	1.6, 0.23, 0.61	1.2x0.46x0.61	Numerical	1(x,y,z,)	$k(x,y,z_1)$	1.89	1.88	1.21	1
90° Gardon radiometer	1.6, 0.23, 0.61	1.2x0.46x0.61 Analytical	Analytical	1047	2.5	1.93	1.88		
Fotal Gardon gage	1.2, 0.23, 0.76	2.4xU.46xU.76	Numerical	1(x,6,2,)	k(x,y,z,)	1.67	2.12	1.30	1
lotal Gardon gage	1, 0.23, 0.70	2.4x0.46x0.76	Analytical	1067	. ć. <u>.</u>	1.71	2.15		
Ray radiometer	1.8, 0.23, 0.23	ŀ	Numerical	$T(x,y,z_1)$	$k(x,y,z_1)$	3.85		1.06	0.64

 $T(x,y,z_1) = 1127 - 50.0 x$ 

 $k(x,y,z_1)=3.028=19.068\ z_1+80.8058\ z_1^2=140.898\ z_1$  Temperature of ceiling and walls in contact with ceiling layer 036 K Wall temperature below ceiling layer:

x is distance from PMMA end-wall [m] y is distance from ceiling side-wall [m]

 $\mathbf{z}_1$  is distance from ceiling [m]

PMMA wail 636 K

All other walls 300 K

Thickness of layer .305 m

# 3.4 USE OF THE APPROXIMATE RADIATION MODEL

When there are significant longitudinal and vertical gradients of temperature and/or absorption coefficient in a ceiling layer, the radiant flux to floor level can be computed reliably from the analytic model, as demonstrated by the previous examples. An average temperature and absorption coefficient for use in the analytic model are computed from equations (13) and (14) for the axial location in the enclosure corresponding to that of the target.

Normally, data on gas and surface temperatures are readily available but not complete information on infrared absorption coefficient as a function of axial position, x, and height z. It is noted in reference 22 that because the radiative flux calculation is relatively insensitive to the exact value of absorption coefficient, neglecting the molecular radiators, CO<sub>2</sub> and H<sub>2</sub>O, introduces little error. Consequently, it should be sufficient to assume that the spacial variation of absorption coefficient within the layer is primarily due to the dilution by ambient temperature air of constant density soot particles. This dilution effect on absorption coefficient can be related simply to gas temperatures by the following relation, explained in reference 22:

$$k(x,z) = \left(\frac{T_o}{T(x,z)}\right) \left(\frac{T(x,z) - T_{\infty}}{T_o - T_{\infty}}\right) k_o$$
 (17)

where  $k_0$  and  $T_0$  are the measurements of gas absorption coefficient and gas temperature at a single, reference location and  $k(\mathbf{x},\mathbf{z})$  and  $T(\mathbf{x},\mathbf{z})$  are the corresponding quantities at any other axial or vertical position in the layer. This relation will only be valid if total gas pressure is constant (the enclosure is ventilated), gas specific heat is constant and the observed temperature distribution is due to entrainment of cold ambient air into the layer. Significant convective or radiative heat losses to the ceiling, side-walls, or to floor level from the bottom of the layer will thus tend to introduce errors in the k(z) obtained from equation (17). Use of equation (17) allows the distribution, k(z), to be calculated from a single measurement of infrared absorption coefficient and plentiful measurements of gas temperature.

## IV SUMMARY OF RESULTS

Experiments with full-scale and model ceiling channels exposed to a developing PMMA wall fire yield three primary results:

- 1. Pressure modeling predictions of flame spread rates under a PMMA ceiling, flame lengths under an inert ceiling and radiant heat loss from the gas layer under both types of ceilings, based on the results of model tests at elevated pressure, are in reasonable agreement with measurements obtained at one atmosphere (full-scale). The behavior of three aircraft material ceilings is not pressure modeled well since fire spread is complete at elevated pressure but does not occur at one-atmosphere.
- 2. A simplified heat transfer analysis of the fire spread process shows that pressure modeling is likely to be more successful when applied to thin flame zones. For this reason, modeling predictions of fire growth will be less accurate when applied to deep ceiling channels than to vertical walls. One possible cause of modeling inaccuracy for the aircraft ceiling materials is shown by the heat transfer analysis to be the elevated surface temperatures associated with char formation. Another possible cause is shown to be the excess of fuel and thermal insulation available at elevated pressures due to the use of the same material thickness as at full-scale. In contrast, the PMMA and inert ceiling thicknesses are able to be properly scaled in accordance with the modeling requirements.
- 3. Exponential growth factors characterizing fire spread rates, mass loss rates and radiant heat loss are determined for all model ceiling configurations at each of three elevated air pressures. Ceiling materials can be readily grouped according to fire growth hazard with these exponential factors, with such grouping being roughly independent of pressure level.

A comprehensive analysis of radiation heat transfer from near-ceiling gas layers yields three main results:

- 1. An exact, numerical solution technique is formulated for computing the radiant flux toward the floor from hot gas layers generated by an aircraft cabin fire. Arbitrary variations in gas temperature and absorption coefficient in all three dimensions are permitted and radiation from the bounding wall and ceiling surfaces is taken into account.
- 2. Simplified analytic and numerical approximations are developed by assuming that only gas radiation properties along a vertical ray directly above the floor-level target are important. A single gas temperature and absorption coefficient for use in an analytic radiant flux expression are obtained from a suitable spatial average of this vertical property distribution. Such an approximation is tested and found to be capable of closely reproducing the exact numerical results, even when sharp, unrealistic changes in gas layer properties are assumed.
- 3. The radiation analyses are generally consistent with the experimental data on full-scale ceiling layer temperature, absorption coefficient and radiant heat loss. Predicted radiative enhancement of the full-scale PMMA wall fire by ceiling layer heat is also consistent with the measured PMMA mass flux distribution.

#### V CONCLUSIONS

- 1. The pressure modeling technique can be used to make conservative predictions of tire growth on walls of under cellings as long as full scale tlame conescite sufficiently thin. The product of absorption coefficient and mean beam length at one atmosphere should be less than about 0.2.
- 1. Tests at clovated air pressure can be used to rank materials according to rate of tire growth in wall or celling configurations as determined from measurements of flame spread rate, mass loss rate and radiant hear loss. Such ranking should be valid as long as results are reasonably independent of absolute air pressure in the range of 10 of atmospheres.
- 3. A simplified, analytic approximation involving the uso of a suitable averaged gas temperature and absorption coefficient should be adequate to: the calculation of tadiant this to their level from ceiling gas lavers with large streamerse gradients in gas radiation properties.

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TABLE A-1. UPWARD FLAME SPREAD ON PMMA WALL TEST 6-10-80, INERT CEILING

Time [s]	Pyrolysis Height [cm]	Flame Height [cm]
192	10.5	21.0
292	14.5	23.5
345	16.0	31.0
380	20.0	33.0
461	26.0	37.0
476	28.0	46.0
505	30.0	46.0
531	32.0	53.5
561	38 <b>.</b> 0	48.0
615	38 <b>.</b> 0	53.0
661	44.0	62.3
658	41.0	65.3
696	42.0	60.3
724	46.0	66.3
761	53.7	77.3
767	55.3	72.3
781	55.8	71.3
791	υC.3	82.4
797	61.3	82.4
80 ಚ	62.3	82.4
819	63.3	82.4
826	65.3	78.3
844	67.5	82.4
85.	60.5	82.4
859	09.3	
863	70.5	
869	72.3	
874	71.3	
899	75.3	
000	76.3	
de	79.3	

Exponential Crowth Factors: Fyrolysis Height  $2.651 \times 10^{-3} \text{ [s}^{-1}\text{]}$ Flame Height  $2.100 \times 10^{-3} \text{ [s}^{-1}\text{]}$ 

TABLE A-2. UPWARD FLAME SPREAD ON PMMA WALL

TEST 6-19-80, INERT CEILING

Time [s] 61	Pyrolysis Height [cm] 9.5	Flame Height [cm]
129	11.5	14.5
180	14.0	16.5
214	16.5	18.5
242	17.5	23.0
289	20.0	24.5
301	21.5	26.5
338	24.0	27.5
<b>39</b> 0	28.0	34.5
440	32.5	41.5
480	35.5	43.5
510	39.0	51.5
540	42.5	55.3
575	46.0	60.3
601	49.0	68.8
634	55.3	67.3
647	55.3	71.8
659	57.3	71.3
682	60.3	78.3
700	63.3	82.7
721	66.3	82.7
738	68.5	
764	72.3	
780	75.3	
825	80.3	
867	82.7	

Exponential Growth Factors: Pyrolysis Height  $2.759 \times 10^{-3}$  [s<sup>-1</sup>] Flame Height  $3.001 \times 10^{-3}$  [s<sup>-1</sup>]

TABLE A-3.-MASS LOSS RATE OF PMMA WALL
TEST 6-10-80, INERT CEILING

Height Above Ignition	Mass Flux
[cm]	[g/m <sup>2</sup> s]
7.6	5.75
23	6.37
38	6.88
53	8.13
69	10.20
84	10.77

Extinguishment 1890 s after ignition.

From preceding mass flux data, average mass loss rate per unit width of wall at 945 s after ignition is 7.22 g/ms

Assuming an average width of burning wall of 0.2 m, total wall mass loss rate is  $1.44~\mathrm{g/s}$ 

MASS LOSS RATE OF PMMA WALL TEST 6-19-80, INERT CEILING

Height Above Ignition	Mass Flux
[cm]	[g/m <sup>2</sup> s]
7.6	7.21
23	7.66
38	8.23
53	9.37
69	11.60
84	12.70

Extinguishment 3577 s after ignition.

From preceding mass flux data, mass loss rate per unit width of wall at 1788 s after ignition is  $8.52~\mathrm{g/ms}$ 

Assuming an average width of burning wall of 0.3 m, total wall mass loss rate is  $2.55~\mathrm{g/s}$ 

TABLE A-4. FLAME EXTENT UNDER INERT CEILING

TEST 6-10-80

Time [s]	Flame Length [cm]
1000	3.0
1050	0.0
1140	6.0
1200	6.0
1262	12.2
1519	21.4
1380	21.4
1413	24.4
1449	36.6
1470	42.7
1500	30.0
1530	33.6
1546	42.0
1549	36.6
1554	27.5
1564 1587 1594	36.6 39.4 45.8 42.7
1617 1627 1650 1680	36.6 61.0 45.8
1713	39.4
1750	51.9
1774	48.8
1800	55.7
1825	63.0

TABLE A-5. FLAME EXTENT UNDER INERT CEILING

TEST 6-19-80

Time	Flame Length
[s]	[cm]
<b>102</b> 0	4.0
1080	10.0
1142	9.0
<b>120</b> 0	15.0
1269	15.0
1320	20.0
1380	25.0
1440	25.0
1470	40.0
1514	40.0
1560	40.0
1622	50.0
1680	40.0
1735	50 <b>.</b> 0
1786	60.0
1788	55.0
1800	50.0
1841	60.0
1861	60.0
1894	65.0
1925	60.0
1950	55.0
1980	70.0
2003	75.0
2040	80.0
2100	98.0
2160	80.0
2221	68.0
2267	70.0
2325	80.0
2498	75.0
2556	128.0
2594	148.0
2664	120.0

TABLE A-6. FLAME EXTENT UNDER NAFEC #234 CEILING

### TEST 7-9-80

Time	Flame Length	Time	Flame Length
[a]	[cm]	[s]	[cm]
1143	0.0	1759	52.0
1232	18.0	1760	62.0
1271	18.0	1761	42.0
1272	24.0	1801	42.0
1324	24.0	1802	42.0
1358	30 <b>.</b> 0	1803	40.0
1359	35.0	1804	42.0
1360	35.0	1846	42.0
1394	35.0	1849	62.0
1398	42.0	1859	42.0
1461	20.0	1860	42.0
1462	35.0	1928	50.0
1463	32.0	1929	55.0
1464	28.0	1930	49.0
1502	26.0	1931	43.0
1503	42.0	1932	57.0
1528	30.0	1992	57.0
1529	65.0	2002	42.0
1531	42.0	2003	67.0
1533	67.0	2004	55.0
1543	67.0	2132	57.0
1567	67.0	2133	60.0
1568	42.0	2134	56.0
1582	54.0	2156	56.0
1589	44.0	2137	55.0
1606	50.0	2198	57 <b>.</b> 0
1609	67.0	2199	60.0
1617	32.0	2200	56.0
1619	42.0	2201	67 <b>.</b> 0
1628	34.0	2240	55.0
1629	42.0	2242	57 <b>.</b> 0
1630	44.0	2287	56.5
1631	54.0	2288	60.0
1632	42.0	2290	67 <b>.</b> 0
1663	34.0	22.30	01.0
1664	35.0		
1665	37 <b>.</b> 0		
1666	42.0		
1667	34.0		
1690	42.0		
1691	48.0 48.0		
1692	42.0		
1693	44.0		
1694	44.0 44.0		
1724	44.0 42.0		
1725	42.0 42.0		
	44.0 44.0		
1726 1727	44.0 42.0		
1758	42.0		

TABLE TABLE A-7. FLAME EXTENT UNDER NAFEC #B8811 CEILING

### TEST 7-22-80

Time	Flame Length	Time	Flame Length
[s]	[cm]	[s]_	[cm]
960	0.0	1727	65.0
1063	10.0	1730	77.0 70.0
1127	5.0	1750	70.0
1135	15.0	1751	68 <b>.</b> 0
1140	15.0	1752 1753	46.0
1200	15.0	1803	77.0
1225	15.0	1805	65.0
1264	25.0 30.0	1807	65.0
1297 1307	32 <b>.</b> 0	1853	58.0
1315	45.0	1854	70.0
1321	30.0	1855	77.0
1336	46.0	1856	80.0
1337	30.0	1874	77.0 71.0
1338	30.0	1875 1876	65.0
1353	32.0	1877	77.0
1354	32.5 35.0	1878	80.0
1366 1367	30 <b>.</b> 0	1879	82.0
1378	45.0	1916	60.0
1380	35.0	1917	62.0
1382	39.0	1918	63.0
1383	33.0	1919	77.0
1405	45.0	1948	70.0 75.0
1406	38.0	1949 1950	72.5
1407	49.0	1951	77.0
1407	46.0 46.0	1987	58.0
1420 1421	44.0	1 <b>98</b> 8	93.0
1422	49.0	1989	65.0
1432	46.0	2059	65.0
1446	48.0	2060	77.0
1447	52.0	2061	90.0
1505	46.0	2062	65.0 77.0
1525	46.0	2085 2086	77.0
1548	55.0	2087	77.0
1551	48.0 58.0	2088	65.0
1552			_
1553 1577			
1580			
1619	_		
1634	_		
1667	48.0		
1668			
1691			
1692			
1697			
1694	1 05.0		

TABLE A-8. FLAME EXTENT UNDER NAFEC #223 CEILING

TEST 7-30-80

Time [s] 1089 1118 1140 1175 1176 1199 1201 1246 1247 1248 1281	Flame Length [cm] 0.0 7.6 7.6 15.2 10.2 12.7 15.2 17.8 20.3 12.7 17.8	Time [s] 1550 1551 1572 1613 1614 1615 1632	Flame Length [cm] 50.8 50.8 58.4 61.0 53.3 50.8 50.8
1282 1284 1299 1301 1314 1315 1323 1328 1335 1366 1368	15.2 17.8 15.2 15.2 7.6 15.2 17.8 20.3 12.7 15.2 17.8 20.3 20.3		
1369 1389 1389 1440 1444 1448 1448 1450 1484 1489	17.8 20.3 22.9 17.8 30.5 27.9 30.5 30.5 30.5 27.9 43.2 43.2		
1494 1495 1509 1510 1511 1512 1546 1549	45.7 40.6 40.6 45.7 43.2 45.7 61.0 48.3 53.3		

TABLE A-9. FLAME SPREAD UNDER PMMA CEILING

TEST 8-5-80

Time	Flame Length
[s]	[cm]
971	0.0
1091	10.0
1105	15.0
1115	30.0
1116	40.0
1128	60.0
1132	70.0
1133	70.0
1139	80.0
1143	90.0
1144	90.0
1156	115.0
1157	125.0
1158	128.0
1159	140.0
1160 1161	130.0
1164	128.0 150.0
1165	130.0
1167	130.0 130.0
1168	140.0
1173	130.0
1173	135.0
1174	140.0
1176	145.0
1177	150.0
1178	140.0
1179	140.0
1188	160.0
1189	160.0
1190	220.0
1197	230.0
1198	240.0

Flame Length Exponential Growth Factor = 0.0249 [s<sup>-1</sup>] (Regression Coefficient = 0.89)
Flame Length Linear Crowth Rate = 1.95 [cm/s] (Regression Coefficient = 0.92)

TABLE A-10. TEMPERATURE AND RADIATION MEASUREMENTS

	W.CC/W	00	00.	00.00	9.	0.00	00.0	00.0	33.6	0.02	0.00	00.00		60.	2	G.	6.	00	G	6.	S.	ος.	00.	: '	00.	o.	0.005	င်	o o	00.	9	0	<u>ن</u> (	٠		. ( . (	٠.	00.	00	ç,	0	8	်	õ	5	9.
	D4 W/C! OM	دغا	10	ď,	ĸ,	۲.)	L)	0.5	r.)	0.0	٠. در	.n.	ربا د	<u>ن</u> 0	о Э	с. m	en.	о .п	6	0	0.0	n,	0.0	0	ο (i)	iu.	\$50.0×	о п	ر. دن:	ம் ! ப	u! O		0 (2)	3) I	ນ (	ກ ແ ວ ດ		v.	n) i	0.5	о	4.0	o 4	٥.	9.0	4
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	52 //C: 005R	ر.	0	ن.	٥.	٥.	ن.	Ú.	Ċ.	φ,	ن.	ر،	٠.	ပ.	٥.	ر)	٥.	٠.	٠.	· .	0	٠	٠.	0	()	Ċ,	-0.001	٠;	ن.	0	٥.	ပ		•	٠ د:	د د	•	0	0	c,	٠.	٥.	۰.	٥.	0	0
NG	E1 /cromsR W	0	Ċ.	8	8	CS.	÷	00.	Ċ.	0	<u>ن</u>	0	٠. د	Ö	0.00	۲̈.	₹.	8	ζ.	0	Ö.	Ċ.	S	0	8	ეე.		$\overline{C}$	S	() (	0	G (	، رن ه رن	٠ ا	3	: . ز د	)	٠,	0	٠ د	<u>ت</u>	3	<u></u>	ွှ	8	8000
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-80. INERT	A11 DES C																										L.)																			152.6
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SCALE TEST	A 9 DEG C		3	Ċ		3,	÷	Ŀ	ċ	Ö	<del>,</del>	<u>.</u> .	-	ç.	Ç.	٠,	٠.	1	c i		θ.			ci	ó	e,	80.8	ო	7	თ	ċ.	4.	: :	٠.		ກ. ທ່າ	ė.	ic.	σ.:	~ ~	ċ	ئ	ċ	'n	ω,	
FULL-S	A 8 DE3 C	ζ,	٠ م	ci	ď	Ö	'n	en.	• r	ς:	'n.	٠.	ζ.	œ.	о О	ö	0	.;	(2)	7.	₹7	.:	5)	io.	ç-	Θ.	40.3	ı,	:0	<u>.</u>			;	- (	5 t	٠,	-	انٽ	٠,	· (i)		ဗ	5.	6.	27.	ė.
	A 7 DES C	22.1	21.1	21.4	22.3	22.9	23.5	24.4	2 : . 2	20.4	8	27.9	٠ ٢٧	30.1	31.8	33.1	35.6	38.2	37.0	33.1	51.3	44.5	43.9	47.9	<b>4</b> . € 3	51.0	54.8	56.7	න : වේ	2.1.0	σ : ω :	T (0	101	n	9 c	01.7	) i	CF (	$\circ$	$\circ$	1:1.	•-	C4	m	57	4
	A 6 DEG C	۲,			ci	ç.	Ġ		٠,	Ġ	ė	۲-	ώ.	٠,	÷.	ć	٠,	ń	ť.	ω,	ç.	ω.	'n.	۲.	ď:	ų.	53.1	٠.	<del>ი</del> .		'n.	<b>بر</b>	Ni t	٠,	· (	,					103.7	(i)	21.		ώ.	126.2
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TEST 6-10-80

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1.2 C) 1.5 1.4 1.4 1.01 1.4	N CH N C N		1	433.

TABLE A-11. TEMPERATURE AND RADIATION MEASUREMENTS

					FITT-	SCALE	TEST 6-19	-80,	INERT CEI	CEILING				
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DS W/CVCM	0.022 0.023 0.025	220	.03	4	.04	53.	.05	0.0	.07	.07	80.	80.0	5 0	2	Ξ	. 12	€ ₹	. 4	19	16	.17	.17		20	.20	2	. 22	. 23	6	9	2.	250	, 0	55
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B1 /CMCMSR W	0.010	2.2.8	2.0.0	.02	69.		9	200	9	0.	0.	2.5	ວິດ	.03	.07	. CB	9.0	0 0	0	- 2	5	2:	- :	2	. 13	.13	<u>+</u>	. 3	7	4 ,	٠. د			.17
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A 14 DEG C	93.9 93.3 103.0		· - «	35.	 		52.	٠.	69	71.	. 0	cy r		(%)	÷	<u>.</u>	<u>က</u> မှု			=	46.	S :	ດ ເ	9 77	57.	.02	74.	9	92			ກຸບ ລຸດ	000	Š.
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<b>2</b>				***				
C3 1/M	ത്ത്ത്		000	1.08	1.12			
B2 /C::CMSR	4 4 7 7 3	2222			4 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	30.4.0.4.4	4 0 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	00000000000000000000000000000000000000
B1 //CMCMSR W/	20040	26240	910 9 9 12 12 10 10	8.38.88	C. E. E. E.		C. C	00000000000000000000000000000000000000
A12 DEG C W	260.	-0000	0 6 6 00	0 5 5 6 6	40-0	0000000	477	785.3 780.8 780.8 780.9 776.1 780.0 778.0 729.5
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A 6 DEG C	6: 10 4: 40 4: 40 4: 10	24.04.0.7.2	6655	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	က် က <b>က ဝ</b> က က က <b>ဝ</b>	- 4 0 7 C C	77.000.000	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
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DS W/CCM	0.151	0.133	0.120	0.108	0.101	0.093	0.035	0.032	0.077	0.072	0.063	0.065	0.061	0.059
D4 W/CMCM	0.417	0.385	0.357	0.330	0.297	0.280	0.274	0.255	0.253	0.225	0.239	0.238	0.227	0.220
C3 1/M	0.45	0.35	0.24	0.18	0.10	0.04	-0.01	-0.04	-0.02	-0.05	-0.07	-0.13	-0.17	-0.18
B2 W/C'CMSR	0.115	0.105	0.088	0.075	0.076	0.065	0.035	0.000	0.055	0.041	0.037	0.048	0.043	0.028
B1 W/CWCMSR 1	0.039	0.005	0.078	0.073	0.071	0.064	0.060	0.033	ۍ0.0	0.083	0.045	0.043	0.041	0.033
A12 DEG C 1	430.7	391.8	352.1	333.7	318.5	302.8	290.8	275.2	203.4	254.7	243.9	241.8	237.7	230.2
A11 DEG C								137.3						
A 10								37.0						
A9 050 0								ტ. ენ						
A8 DEG C								64.0						
A 7 DEG C								142.2						
A 6 DEG C														166.3
A 14 DEG C	271.6													
A 13								37.3						
TIME	2751.8	. 2782.2	. 2002 .	. 2823.0	2843.4	. 2633.7	. 2884.0	- 2004.4	. 2924.7	. 2045.0	. 2985.3	. 2935.6	3006.0	3026.3

TABLE A-12. TEMPERATURE AND PADIATION MEASUREMENTS FULL-SCALE TEST 7-9-80, NAFEC #234 CEILING

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A11 °C	, ر	- 4 0 0 0	ი	(1)	ъ.	ເກ	'n	ö	۲.	: 5	9.	ပ	<del>.</del> .	'n	έ.	Ġ	Ġ	'n	ö	<b>.</b> :	'n	ທ໌	ú	'n	ó	ë,	4	۲,	'n	'n	'n,	æ	e,	щ.	(D	'n	۲۰	÷	ر. د	ci.	Ċ	4,	œ.		135.4
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A6 °C																							ö	6	φ.	ż	ë	Ġ	m.	ċ	4	œ.	÷	'n.	;	ż	ώ.	۲.	ú	'n	۲.	ë.	118.7	ď	
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D4 W/CMCM	ć	. 6	9	.02	.02	.03	8	S	9	9	.04	.04	90.	. 25	.00	.07	.08	.09	Ξ	7	.13	.16	.17	.13	<u>.</u>	. 23	. 22	.23	, 20	<u>ښ</u>	. 33		., c	0.040		, C.	37	99		6	41	4.2	. 42	4
C3 1/M				•	•	٠	•	•	•		٠	٠	•	٠	•	•	•	•	•	٠	•	•	•	•	•	•	•	•	٠	•	•	•	•	3.0					٠	•	•	•	•	
B2 //CMCMSR	5	0	0	8	9	0.	.02	9	ç.	.02	9	. 02	.02	9	9	.02	9	.02	.02	.02	0	.04	.04	.04	.05	.05	.03	.05	90.	.00	.0	56		0.00	Ċ	0.7	03	0.	.08	9.	.08	.08	.08	60.
E1 I/CNUSSR W	Ċ	9 0	0	0	00.	00.	00.	CJ.	ö	00	5	5	٥.	0.	9	6	6	.02	. 62	. 62	.02	. g3	.03	.03	.03	.03	9	.04	.05	00.	90.	3.6	9 6	0.0	• E		0.0	0.0	.07	.07	.07	.07	.07	0.
A12 DEC C				В	9	Ċ.	39.	87.	39.	33.	32.	5	34.	29.	43.	51.	33.	.7.	22.	5.	21.	27.	49.	40.	63.	94.	00	07.	35.	93.	22	4.4	- c	1 6	9		75.	ເກ	93.	85.	79.	<u>.</u>	00	21
A11 DEG C	c		;	ω.	ი	71.	79.	89.	91.	5.	0.2	<u>.</u>		23.	8	80.	040	۲.	26.	٠ ۳	46.	53.	0.	ω ω	00	23.	-	43.	9e	ω ()	000		, c	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	ο σ	 	 	ເນ ເນ	68	94.	95.	3.	05.	24
A10 DEG C	-				<u>.</u>	ų,	ċ	ö	'n	÷	4	ë	m	4	ທ່	œ.	ທ.	۲.	ω̈.	i,	ω.	Ċ	÷	'n	ö	ū.		'n		٠,	<u>.</u>	n	ກຸມ	0.0		, σ	'n	ä	8	6	8	÷	÷	
გ გ ე ე∃ი	α			щ	θ,	ä	7.	თ	œ,	(j)	ω.	Ġ	'n	о О	'n	ä	ö	ż	5	Ġ	4	ú,	÷	'n	'n	რ .		ໝ	u) i	φ,	4.	n o	0 u	75.6	c	4	6	0	ö	Ť.	7.	.;	ů.	113.5
A B DEG C																78.	4.	:	20.	<del>1</del> 6.	05.	37.	80.	gc.	95.	96	ဗ		ე		33	. Y	, , ,	361.0	7.	52.	4.	52.	54.	69	75.	9.	88.	99.
A 7 DEG C	σ		2	4	છ	7.	ر. دي	83.	87.	97.	03.	50.	19.	10	39.	55.	75.	98.	39.	<u>-</u>	29.	54.	58.	79.	77.	78	-	27	(D)	(1)	ις 1 C4 (	D (	מים	257.3	4	58.	20	54.	57.	7.	76.	75.	73.	78.
A 6 DEG C	4	0		e,	į,	77.	83.	93.	94.	ое.	<u>6</u>	6	59.	28.	ů.	93.	69.	23.	41.	50.	Ų	<u>.</u>	ij.	17.	ig.	0	er.	62.	34.	0	99		- u	400.00	0.0	77.	98	9.	09.	90.	95.	8	2.	60
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C3 1/≅	44440	ញ្ <b>ល់ជុ</b> ម្មិលមើល្ខាស់ស	արտարագա	0.0000000000000000000000000000000000000	000000000
B2 //CMC/YSR	80.00	20-2-2-2-2	2447446	00000000000000000000000000000000000000	5000000000000000
S. S				00000000000000000000000000000000000000	000000000000000000000000000000000000000
A12 DEG C 1	67.57	6 600000m	200.000.000.000.000.000.000.0000.0000.	0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000 0.000	- move on - co o do d
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A 7 DEG C	0 - 9 0 5 0 0 0 0 0 0	7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	12 0 0 0 1 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	$\frac{1}{2}$	
A 6 DEG C	95. 14. 28.	89.7-7-89.69	34.35.35.35.35.35.35.35.35.35.35.35.35.35.	0.00111 0.00111 0.00111 0.0010	1010-040000000
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D5 W/C!ZCM	00000000000000000000000000000000000000	
D4 W/CMCM	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0	
C3 ≠/≠	000000000000000000000000000000000000000	
R2 W/CMCMSR	00000000000000000000000000000000000000	111
B1 W/CMCNISR W	000000 000000 0000000 0000000000000000	
A 1.2 DEG C	844300400004444 8643004000044444 46040000000000000000000000	
A11 DEG C	<i>ผลข</i> อยผูชผูชสุดผู <b>ช</b> เกราะ (การสุดผูช เกราะ (การสุดผูช (การสาดผูช	
A10 DEG C	4446 4446 4446 4446 4446 4466 4466 446	
A 9 DEG C	23333333333333333333333333333333333333	
A B DEG C	22 22 22 22 22 23 24 24 26 26 27 27 27 27 27 27 27 27 27 27 27 27 27	
A 7 DEG C	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	
A 6 DEG C	0.08 0.08	
1ME SEC	2791.2 2811.8 2812.4 2812.4 2813.9 28854.1 28918.6 29918.3 29918.3 29918.3 29918.3 30918.5	

TABLE A-13. TEMPERATURE AND RADIATION MEASUREMENTS

	D4 D5 /CMCM W/CMCM		00.0 100.	.001 -0.0	.001 -0.00	.001 -0.00	.001 -0.00	.001 -0.00	000- 000	.001 -0.00	.002 -0.00	.001 -0.00	.002 -0.00	.002 -0.00	.001 -0.00	.001 -0.00	.001 -0.0	.001 -0.00	.001 -0.00	.003 -0.00	.004 -0.00	.003 -0.00	.004 -0.00	.002 -0.00	.001 -0.00	.003 -0.00	00.00	00.00.00	00.0- 500.	00.0	00.0- E00.	50.0	90.0 400.	500	00.0	00.0	.007 0.00	00.0 0.00	00.0	00.0 - 0.00	0.008 0.002	.007 0.00	00.0 800.	00.0 600.	.011 0.00	•
	£ 7 3 ₹	į	٥.	•	9	0	٥.	٥.	٥.	٥.	٥.	٥.	0	0	٥.	°.	0	°.	٥.	٥.	٥.	٥.	ပ	٥.	٥.	•	٥.	0	0	٠	٠			9	. 0	0	0	0	0	٥.	00.0	٥.	•		•	C
CEILING	B2 W/CMCMSR		٥.	٥.	0	٥.	0.0	0.	0.	٥.	0.0	0.0	٥.	0.	0.	0.	٠.	0.0	0.0	9	٥.	٥.	٥.	٥.	٥.	۰.	0.	0	0	9			. c			0	٥.	0	۰.	0	-0.005	٥.	٥.	٥.	٥.	C
#B8811	B 1 W/CMCMSR		0.00	õ	0.0	0.00	ó	<u>.</u>	00.	ô	õ	0.	8	00,	် ၀	00.	8	္ပ	<u>.</u>	0.	ő.	00.	ō.	00.	60.	<u>3</u> .	0.00	0.0	8	8	000	5 6			0.0	0	00	8	6	8	-0.010	8	9	8.	8	Č
-80, NAFEC	A12 DEG C	, i	'n.	26.3	Ġ	7	7	0	œ,	ö	ö	ö	ċ	ė	ю Э	4.	'n	ė.	ζ.	ò.	ċ	ä	'n	4.	ώ.	ä		·	ċ	N I	٠,	ėυ	. a	. 4		0	ര്	ம்	œ.	ë	82.1	Ġ	ä	ů.	ö	ı
7-22	A11 DEG C		9	9	ė.	7.	æ.	ö.	9	ö	ö	<del>.</del> .	5	e,	ლ	4.	'n	ď.	ω.	ó,	÷.	ć	4,	ъ.	7.	7.	œ.	6	<u>.</u>	<u>-</u> ,	٠ د	D C	• -				6	რ	ທ	θ.	83.9	ä	6.	ö	ó.	٥
SCALE TEST	A10 DEG C	i	ď	ċ	е Н	4	щ	ë.	က်	'n	4	<del>ن</del>	ë.	ю :		е Э	Ď,	4.	ë,	S.	4.	Ġ.	÷	4	ر. د	• •	4	4.	m	'n.	<b>1</b> 1		; ~	4	'n	4	4	ω.	4.	7	26.8	4.	4.	4.	4.	4
FULL-S	A 9 DEG C	i	2	e,	4.	٠.	۲,	ίŲ.	ທ່	7.	ė.	4.	2	ů.	щ	7	۲.	7	7.	ω.	9	7.	7	ь. О	თ	φ.	ი .	- 1	٠,	4 (		> c		_	ω,	2	6	Э.	6	。	31.9	ო	6	ო	-	Ö
	A B DEG C	)	e.		'n	'n	ů.	E)	7	ė.	ė.	۲.	۲.	œ		ó	÷	ė	ö	ë.	ä	ė.	ė	۲.	÷	က်	٠. س	٠. ن	ė	٠,		٠,	. 4	9		÷	'n	ė.	Š.	œ.	73.3	9	ė.	'n	4.	c
	A 7 DEG C	1	4	25.1	ທ.	ιΩ	ė.	7	۲.	ж ж	7	σ.	Ö	<del>.</del>	'n	ċ.	'n	4	ę,	7.	ь б	ö	.:	ъ.	4.	ທ	۲.	ش د	s o		ή,	: u	. 0		ά.	9	ė.	<u>.</u>	÷.	3	78.2	÷.	'n	6	ö	œ
	A 6 DEG C	)	4.	4	ů.	ເດ	S.	7	7.	ë	ė.	о О	ö	ဂ	'n	ä	e,	4	'n	7.	œ.	ö	ö	'n	4	Š.	Ċ,	œ (		რი		วัน	· a	σ,	;	'n	7	<u>.</u>	ö	ش	78.2	ö	ö	Ġ	ö	ď
	TIME	)	7.	<del>ر</del>	٠ •	س	'n	œ.	05.	2.	37.	54.	70.	85.	95.	9	35.	51.	63.	54.	00	17.	33.	49.	ω· «	v (u)	8	. 5	- :		• •	ວິທີ ບິດ		9.00	45.	62.	78.	94.	Ξ.	27.	643.8	60.	76.	92.	9	25

	m	4	4	ស	ဖ	7	ന	ന	7	7	_	2	က	4		7	0	~	ო	9	თ	<b>-</b>	~	4	7	80	0	4	60	_	ς.		് ന	-	ო	4	თ	ഗ		9	ហ	ო	6	ო	6	
DS W/CMCM	0	00.0	0	٥.	٥.	Ö	0	٥.	0	٥.	٥.	0	0	٥.	٥.	٥.	٥.	0	٥.	0	٥.	٥.	٥.	٥.	٥.	٥.	0	٥.	٥.	٥.	٥.	٥.	٥.	٥.	٥.	٥.	٥.	٣.	۳.	٣.	۲.	۳.	٠.	۲.	0.14	
D4 W/CMCM	0	0.014	.0	6	0.	5	.02	. 02	9.	.02	.02	.03	.03	.03	9.	.04	.05	.05	.05	.08	.07	.07	.08	.08	.09	٥.	5.	Ξ	12	. 12	4	5.	.16	-18	2	.23	.24	.25	.27	.28	Ę,	.32	.33	.35	35	
c3 1/₹	0	0.01	٥.	0	٥.	٥.	0	٥.	٥.	٥.	٥.	0	٥.	°.	٥.	٥.	0	٦.	°.	۲.	۲.	٥.	٣.	~	٦.	٦.	Τ.	٦.	٣.	۳.	٣.	٦.	Ġ	٦.	?	3		?	3	ď	7	ω,	ຕຸ	ო.	ų,	
B2 W/CMCMSR	0.0	-0.011	0.00	6	80.	٥	8	00.	0.	8	8	°.	8	8	ö	6	9	00.	٥ ٥	٠.	9	6	9	õ	.02	.02	.03	.02	٥.	.02	.02	.03	.04	0.	.04	.04	.05	.05	.06	90.	.07	.07	.07	.08	60.	
B.I. W/CMCMSR	c	-0.007	0.0	0.0	0.0	٥.	0.0	٥.	9	0.0	0.0	0	٥.	°.	٥.	٥.	٥.	٥.	٥.	o,	0	°.	°.	°	٥.	0	9	٥.	٥.	0	°.	٥.	9	٥.	٥.	٥.	٥.	٥.	٥.	٥.	٥.	9	٥.	٥.	9	
A12 DEG C		109.8											70.	88.	99.	12	25.	23.	35.	34.	51.	47.		77.	299.9	Ξ.	28.	27.	332.3	66.	ວິນ.	68.	73.	07.	90.	Ξ	47.	49.	24.	69.	55.	51.	70.	36.	59.	
A11 DEG C	٩	115.4	9	8	9	<del>.</del> :	8	ω.	。	8	CA.	8	6	. 6	98.	06.	15.	24.	41.	32.	59.	58.	5.	65.	83.	85.	86.	14.	21.	12.	30.	42.	64.	90.	20.	22.	48.	44	45.	61.	92.	99.	6.	6	98-	
A10 DEG C	ı,	25.5	ů.	9	5	7.	δ.	ė.	Ξ.	<u>.</u>	3	eg	۲.	7.	ů	۳.	5	ó	ė,	6	œ,	7.	۲.	8	ë.	ó	ė,	ė.	ъ б	4.	÷	<u>.</u>	ė.	'n	æ	ŝ	Ŕ	4.	œ.	e,	ů.	6	۲.	ö	ď	
A 9 DEG C	c	31.6	4.	ö	4.	ö	Θ.	6		Ġ.	ŝ.	ė.	ö	9.	<b>.</b> :	7.	4.	ď,	7.	6	3	ů.	S	73.	ė	7.	8	ė.		3	ċ	ë	7	4.	4.	ω̈.	Ġ.	ä	8	щ.	6	ď	76.	ö	05.	
A B DEG C	ď	98.0	о О	4.	6	5	ö	S.	ø	6	ъ.	7.	<u>ن</u>	ö	ė.	<u>.</u>	ς.	96.	00.	2.	17.	17.	ö	20.	30.	36.	45.	<u>.</u>	59.	.99	80	89	90.	9	37.	13.	38.	29.	33.	39.	56.	64.	64.	71.	79.	
A 7 DEG C	-	105.5	2	4.	ö	ö	4.	7.	ŗ.	4	ö	69.	75.	82.	91.	02.	4.	18.	32.	27.	46.	52.	4.	59.	69.	78.	82.	95.	=	6.	25.	28	44	98	ò,	07.	<del>α</del>	17.	36.	25.	42.	55.	50.	49.	99	
A 6 DEG C	C	105.6	ď	4.	<u>.</u>	31.	33.	38.	41.	52.	55.	ď	71.	82.	3:	40	4.	٠	27.	29.	67	53.	62.	58.	63.	77.	82.	94.	ō.	90	23.	28.	53.	87.	66	22.	31.	27.	44.	34.	61.	68.	71.	55.	67.	
TIME	-	758.2	74.	91.	57.	23.	39.	56.	72.	88.	05.	21.	37.	33.	70.	86.	80	9.	35.	51.	33.	84.	100.	17.	33.	50.	66.	82.	99.	<u>.</u>	33.	ა გ	77.	94.	0	27.	43	00	76.	93.	99.	25.	42.	58.	74.	

DS W/CMCM			idaddad	idaddad	üüüüüüüüüü	00000000000000000000000000000000000000
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C3 1/M	_ ന ന ന ന ന	4 4 4	444400	ម្រំ 4 លេល លេក	មក់សំសំសំសំសំសំសំ	20000000000000000000000000000000000000
B2 W/CMCMSR	80.0	500000	112000		<u> </u>	
B1 W/CMCMSR v	00000		0000-0	?		0.0033 0.0033 0.0033 0.0033 0.0033 0.0033
A12 DEG C	23.0	772. 773. 773. 773. 69.	000000000000000000000000000000000000000	229. 229. 30.	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	699.4 699.4 699.4 699.5 675.7 675.7 675.6 776.5 176.5 156.9
A11 DEG C	922.	24 + 6 4 <b>6 6</b> 6 4 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	748777		0	6644 6644 6644 6644 6644 6644 6644 664
A10 DEG C	0-6-6		. 4 0 0			0 0 0 0 0 0 4 0 4 0 0 0 0 0 0 0 0 0 0 0
A 9 DEG C	73. 994. 41.	. 4 4 6 4 4 6 4 6 4 6 6 6 6 6 6 6 6 6 6	30.0576		70 92 7 39 6 7	
A 8 DEG C	93.		14 6 4 6 8 4 4 6 8 8 4 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	. 6 4 6 7	684. 725. 726. 720.	485.8 485.8 489.6 489.6 488.9 488.9 475.0 75.0 75.0 75.0 664.0
A 7 DEG C	657.		0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		2 2 3 3 3 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	544 544 544 544 544 544 544 544 544 544
A 6 DEG C	669.	0 1 2 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	- moooo	24-60	220. 221. 225. 235.	25.00 25
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o4 ¥∵cros	0.000000000000000000000000000000000000
C3 1/™	001100110010010010010010010010010010010
B2 W/CCCMSR	00000 00000 00000 00000 00000 00000 0000
B1 W/C:50:5R #	000000000000000000000000000000000000000
A12 DEG C V	1322.0 1323.0 1323.0 1223.3 1223.3 1004.0 1004.0 1004.0 1004.0
A11 DEG C	20000000000000000000000000000000000000
A10 DEG C	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6
A G DEG C	80008800000000000000000000000000000000
A B DEG C	4 R 4 W G 4 Q 4 W W G G G W W G G G W W G G G G G G G
A 7 DEG C	1.000000000000000000000000000000000000
A 6 DEG C	227 227 227 207 207 207 207 207 207 207
W O V til V til F	<ul> <li>6</li> <li>6</li> <li>7</li> <li>8</li> <li>9</li> <li>9</li></ul>

TABLE A-14. TEMPERATURE AND RADIATION MEASUREMENTS

71ME SEC	A 6 DEG C	A 7 DEG C	A 8 DEG C	A 9 DEG C	A10 DEG C	A11 DEG C	A12 DEG C	B2 W/CMCMSR	. C3 1/K	D4 W/CMCM	DS W/CMCM	
7.0	22.	ď	_:		20.1			٥.		0.0	٠.	
23.5	22.7	22.8	21.7	20.8	20.3	24.7	23.2	900.0-	90.0-	-0.002	-0.005	
39.9	23.	ė,	ä	ö	20.3	ų.		٠.	•	0.0	٩	
56.4	23.		ë	ö	20.1	<u>ن</u>		٥.	•	0.0	0.	
72.8	24.	4.	ë	ö	21.1	'n.		٠.	•	0	٥.	
89.1	24.	4.	4.	8	20.4	ė.		٥.	•	0.0	٩	
105.6	25.	4.	4	。	20.6	۲.		9	•	0.00	٥.	
122.0	26.	ė.	24.1	<u>.</u> :	20.6			0.003		0.00	0	
138.4	26.	6	4	6	21.7	ä		9	•	0.00	•	
154.8	27.	7	'n.	ω.	21.4	6			P	ö	0.0	
171.2	28.	6	ů.	ω.		6			ò		9	
187.7	29.	8	'n.	-	20.4	6			Ģ		0	
204.1	29.	6	7	5	20.9	ď			ė		-0.0	
220.4		ö	ů.	•	20.9	ď		•	•		9.0	
236.8		ä	7		20.6	4		•		Ö	-0.0	
253.3	•	32.	ö	4	21.6	4		•			0.0	
269.6		4.				ė	37.5		•	0	9.0	
286.1	35.	ŝ	ë	6	21.4	ė.			•	Ö	0.0	
302.5	36.	ė.	ë	ë	22.6	8		•	•	8	0	
318.9	37.	œ.	'n	7	23.3	6		•	•	0.000	0.	
335.2	38.	æ	ó	•	20.7	÷.		•	•	90.	-0.003	
351.6	39.	6	ë	25.9	21.4	٠	45.5		•	0	٥.	
368.0	41.	÷	ë	e,	21.2			•	•	0	٥.	
384.4	43.	ö		•	٠.	•	48.2	•	•	0.0	٥.	
400.8	44.	4.	÷	ы		•		•	•	0	٥.	
417.2	45.	ė.	ë	26.1	21.6		52.6	-0.00	•		٥.	
433.1	. 48.	ω.		'n	22.4	ö	53.7	•	•	9	٥.	
450.1	49.	6	ė	•	23.9	e,	55.1	•	•	°	٥.	
466.5	52.	'n		6	24.5	რ	57.1	•	•	o	٥.	
482.9	53.	4		Ġ		'n		•	•	•	•	
499.3	55.	υ.	ë	о О			61.5	•	•	0	٥.	
515.7	57.	۲.	4	œ 1		٠		•	•	o	٥.	
532.1	58	6		۲.	22.6	က်	63.5	0	•	o.	۰	
548.5	62.	'n.		'n.		'n		•	•	0	٥.	
564.9	64.	വ	_	ė		σ.		•	•	٥.	٥.	
581.4	. 99	ຜ່		တ်	22.0	o.		ó	ö	9	•	
597.8	99	σ.	4	ö		÷		•	•	0	٥.	
614.2	71.	₽.	'n	ė.	22.9	ŝ.		٥.	•	ó	٥	
630.6	73.	ä	œ.	œ	-			9	•	ö	۰.	
647.0	75.	ů.	ું	ö		÷		٥.	•	ó	٩	
663.4	79.	6	4	ö	21.5	ë		٥.	•	0	٥.	
679.8	. 83.	ě	75.9	2	4	Ġ		٠.	•	9	•	
696.2	87.	ė.	0	30.4		÷.	90.5	0	-0.00	0	0	
712.6	90.	6	81.1	31.2	21.6	•		8	•	9	٥.	
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ıo	DEG C	DEG C	DEG C	DEG C	DEG C	DEG C	DEGC	W/CMCMSR	1/1	W/CMCM	M/CMCM	
'n.		- (		30.8	က်	<u>.</u> .		5.5	- •	600.0	•	
. m	108.1	108.0	97.6	36.1	23.4	112.7	118.0	0.0	0.17	0.010	0.002	
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A12 DEG C		35	575.4	91.	96	15.	69	79.	33.	72.	28.	610.7	79.	07.	90.	648.7	47.	83.	93.	9	76.	77.	0	80	705.7	671.4	9	92	0	n c	9	្ត ស្ត	3.	23.	93.	38.	42.	38.	29.	301.6	34.	400.7	365.6	9
A11 DEG C			515.8							580.1		566.1							600.4		601.4		591.2			621.9					662.5		649.5							208.5				
A10 DEG C		45.6	46.7	9	47.3	9		2	~	4	ë	50.7	ö	ζ.	mi.	4		ö	54.5			0.09	4.		٠.		٠.		90.0	•	60.7						80.9	68.7	55.0	33.0	31.7.	31.8	30.0	4.55
A 9 DEG C	206.1	251.1	5	212.1	205.0	242.9	238.8	185.9	219.7	225.2	237.3	262.5	215.3	261.6	20	261.9	214.9	263.8	247.6	276.0	248.9	283.5	313.5	281.1	266.9	209.5	7.687	4	- 40	276.3	2 8	291.7	23	274.2	283.3	273.3	282.5	62.		44.4	- 4		•	1
A B DEG C	=	8.	422.5	32.	39.	46.	48.	45.	63.	49.	69.		44.	7.	20	28	67.	64.	79.	68.	9	4	67.	-	89				מים	9 4	5	74.	96	80	4.	81.	95.	Ξ.	=	99.1				4
A 7 DEG C		85.	93.	85.	94.	12.	÷	93.	08.	08.	25.	.60	07.	17.	5.	<u>۔</u>	29	25.	32.	24.	23	<del>.</del>	9	50	35.		B		N C	֡֝֜֝֜֜֜֝֜֜֜֝֓֜֜֜֝֓֜֜֜֜֝֓֓֓֜֜֜֜֜֜֜֜֓֓֓֓֡֜֜֜֡֓֡֓֡֓֜֜֜֡֓֡֡֜֜֡֡֡֡֡֓֡֡֡֡֡֡	46	3.	45	43.	58.	51.	45.	55.	76.	190.0	'n	ä		4
A 6 DEG C	56	79.	97.	90	07.	5	9	98.	õ.	13.	2.	22.	9.	17.	<u>.</u>	<u>~</u>	35	33.	25.	<u>.</u>	3	<u>.</u>	4	5	34.	53	4	5	0.0		46	33	4	37.	39.	46.	36.	47.	<del>ب</del>		8	171.9	4	4
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B2 M/CMCMSR	•	•	• •	•	•	•	•	•	•		0.012
DEG C	312.8	292.4	259.1	246.0	233.4	223.1	213.5	£03.5	4.00	183.4	177.1
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DEG C	29.0		29.3	28.2	29.5	27.6	28.6 28.6		20.00	26.2	28.2 27.3
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TABLE A-15. TEMPERATURE AND RADIATION MEASUREMENTS

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ING	B2 W/CMCMSR	-0.004	•	•		a				•	٠		•	•			•	•	•	•			ö	ö	ö	<i>.</i>	•	; ;	•	•	•		•	•	30	8	5	0	8	0.012	٠
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8-5-80, P	A12 DEG C												26.0				27.3	29.1	29.4	29.2	. כ אורי	:	Ξ.	30.8		32.2	32.1	34.0	33.1	33.4	36.1	36.2	37.1	28.3 4.05	- 9 - 9 - 9	38.7	38.2	39.1	40.4	40.8	
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TABLE A-16

# TEMPERATURE AND RADIATION MEASUREMENTS

## FULL-SCALE TEST 8-20-80

## FULLY DEVELOPED PMMA CEILING FIRE

Instrument	Measurement	
1.D.	Average Stan	dard Deviation
A6	633°C	23°C
A7	708°C	32°C
Α8	782°C	14°C
Α9	847°C	29°C
A10	764°C	19°C
A11	757°C	26°C
A12	793°C	5.7°C
В2	0.643 W/cm <sup>2</sup> sr	$0.027 \text{ W/cm}^2 \text{sr}$
С3	3.51 m <sup>-1</sup>	$0.02 \text{ m}^{-1}$
1)4	$2.122 \text{ W/cm}^2$	0.063 W/cm <sup>2</sup>
1)5	$1.882 \text{ W/cm}^2$	0.040 W/cm <sup>2</sup>